

Problem #1 - 20%

1. True or False:

- The **Finite Element Solution** approaches the exact one as the aspect ratio approaches zero.
- The **Tetrahedral** element is suitable for bodies with **complex boundaries**.
- Both the principle of **virtual work** and that of **minimum potential energy** are applicable for linear-elastic material behavior.
- Equilibrium of the whole structure is usually satisfied; **right or wrong and Why?**
- The **Finite Element Solution** bounds the **exact** one from above?
- The degrees of freedom per node for the **3-D** element are u and v .
- The **LST** element is not generally preferred over the **CST** element.
- Transformation** is not necessary for **frame elements**.

2. Sketch the shape function **N3** for the beam element.

3. Sketch the relation between **Lower-bound, Upper-bound, and Exact solutions**.

Problem #2 - 20%

Use anti-symmetry to simplify the beam in Fig. 1. Then, it is required to determine the nodal displacements (δy and ϕ at Nodes A, B, and C), sketch the deformed shape, and draw the **SF** and **BM** diagrams.

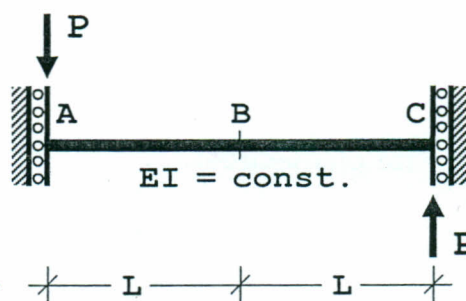
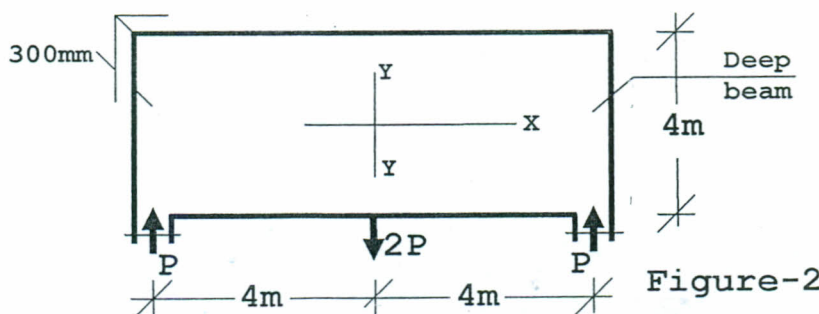


Figure-1

Problem #3 - 20%

For the **bottom-loaded simple deep beam** shown in Fig. 2, it is required to:

- Specify the relevant **type of analysis** to be considered.
- Specify the appropriate **element type**.
- Draw the **load path (strut-and-tie model)** and, consequently, **graph** the stress variation σ_x along the line **y-y**.
- Use symmetry to **simplify** the beam model and, then, apply the **boundary conditions**.
- Discretize** the beam with the appropriate **mesh**.

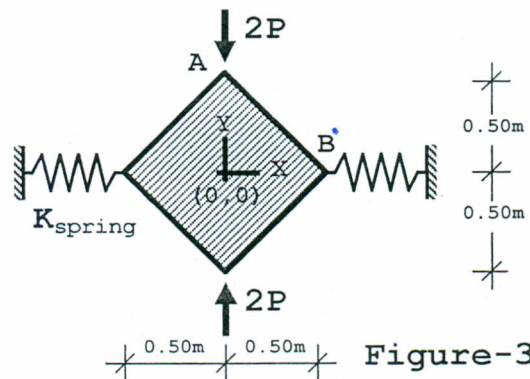


Problem #4 - 40%

For the **thin steel plate** shown in Fig. 3, it is required to:

1. Specify the relevant **type of analysis** to be considered.
2. Specify the appropriate **element type**.
3. Determine the **vertical displacement** at node **A**, the **horizontal displacement** at node **B**, and the **average stresses** within the **body** of the plate under the effect of the given **load** only when:
 - a. K_{spring} approaches **zero**; i.e., **soft** element.
 - b. K_{spring} approaches **infinity**; i.e., **rigid** element. Then, **comment on your results**.
4. Draw the **deformed shape**.

Given: $E = 200000\text{N/mm}^2$, $t = 20\text{mm}$, $\nu = 0.30$, and $P = 100\text{kN}$.



$$[D] = \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix} \frac{E}{(1-\nu^2)} \text{ for plane-stress}$$

$$[D] = \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix} \frac{E}{(1+\nu)(1-2\nu)} \text{ for plane-strain}$$

Best wishes
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