

MATHEMATICAL SIMULATION OF  
MACHINE TOOL STRUCTURES

BY  
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1 - SUMMARY:

Machine tool structure is the principal part of the machine which contributes to its dynamic behaviour. For this reason, the study of the different alternative design configurations of the structure will be necessary, and the mathematical simulation of such structure becomes an effective tool.

In this work, the dynamic characteristics of the structure of a horizontal knee-type milling machine are investigated. Basing upon the classical beam theory, a mathematical model for the structure is introduced. With the aid of this model, fourteen suggested design solution are studied and compared with each other. The suggested fourteen structures are of the same height, main dimensions, wall thickness and mass.

The study results in predicting the most suitable structure. Thereby a considerable improvement of the dynamic behaviour of the considered machine is achieved.

2 - INTRODUCTION:

Due to the development of digital computers, several analytical techniques have been adapted for predicting the static and dynamic behaviour of machine tool structures. In fact, these techniques are based on simulating the structure of the machine by a mathematical mode. Thereby

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the different influencing parameters could be inter-related and their effects could be investigated. Applications of such techniques to some types of machine tools are found in the literature<sup>(1-9)</sup>.

In this work, one of these techniques, namely the classical beam theory, is used to investigate the effects of the structural configuration on its dynamic behaviour. The structure of the horizontal knee-type milling machine is chosen, and the different possible variations in its shape are studied.

3 - THE MATHEMATICAL MODEL:

Fig. (1) shows the general shape structural main parts and the main dimensions of the considered horizontal knee-type milling machine. The basic model of the machine is shown in Fig. (2). To adapt such structure to beam equations the structure should be replaced by an idealized model Fig. (3), which is in turn transformed to the mathematical model shown in Fig. (4).

4 - DERIVATION OF THE FREQUENCY EQUATION:

As shown in Fig. (4), the mathematical model of the machine comprises five beams. Accordingly the general solution of the beam equation (10) could be written in the following general form:

$$y_r = C_{1r} \sin \frac{z_r}{L_r} x_r + C_{2r} \cos \frac{z_r}{L_r} x_r + C_{3r} \sinh \frac{z_r}{L_r} x_r + C_{4r} \cosh \frac{z_r}{L_r} x_r \dots\dots\dots (1)$$

Substituting the end conditions of the different beams into equation (1) yields:-

$$C_{21} + C_{41} = 0 \dots\dots\dots (2)$$

$$C_{11} + C_{31} = 0 \dots\dots\dots (3)$$

$$\sin z_4 C_{14} + \cos z_4 C_{24} - \sinh z_4 C_{34} - \cosh z_4 C_{44} = 0 \dots\dots\dots (4)$$

$$\cos z_4 C_{14} - \sin z_4 C_{24} - \cosh z_4 C_{34} - \sinh z_4 C_{44} = 0 \dots\dots\dots (5)$$

$$\sin z_1 C_{11} + \cos z_1 C_{21} + \sinh z_1 C_{31} + \cosh z_1 C_{41} = C_{22} + C_{42} \dots (16)$$

$$= m_5^2 (\sin z_2 C_{12} + \cos z_2 C_{22} + \sinh z_2 C_{32} + \cosh z_2 C_{42}) \dots (15)$$

$$E I_2 a_2^2 (-\cos z_2 C_{12} + \sin z_2 C_{22} + \cosh z_2 C_{32} + \sinh z_2 C_{42}) - D_{25} C_{25} + D_{25} C_{45} = 0 \dots (14)$$

or

$$E I_2 a_2^2 (-\sin z_2 C_{12} - \cos z_2 C_{22} + E \sinh z_2 C_{32} + \cosh z_2 C_{42}) = E I_5 a_5^2 (-C_{25} + C_{45})$$

$$\cos z_2 C_{12} - \sin z_2 C_{22} + \cosh z_2 C_{32} + \sinh z_2 C_{42} - B_{25} C_{15} - B_{25} C_{35} = 0 \dots (13)$$

which could be written in the form:

$$a_2 (\cos z_2 C_{12} - \sin z_2 C_{22} + \cosh z_2 C_{32} + \sinh z_2 C_{42}) = a_5 (C_{15} + C_{35})$$

$$C_{25} + C_{45} = 0 \dots (12)$$

$$+ \sinh z_3 C_{33} + \cosh z_3 C_{43} = E I_4 a_4^2 (-C_{14} + C_{34}) \dots (11)$$

$$E I_3 a_3^2 (-\cos z_3 C_{13} + \sin z_3 C_{23} + \cosh z_3 C_{33} + \sinh z_3 C_{43}) = E I_4 a_4^2 (-C_{24} + C_{44}) \dots (10)$$

$$E I_3 a_3^2 (-\sin z_3 C_{13} - \cos z_3 C_{23} + \sinh z_3 C_{33} + \cosh z_3 C_{43}) = a_4 (C_{14} + C_{34}) \dots (9)$$

$$\sin z_3 C_{13} + \cos z_3 C_{23} + \sinh z_3 C_{33} + \cosh z_3 C_{43} = C_{24} + C_{44} \dots (8)$$

$$\cos z_5 C_{15} - \sin z_5 C_{25} - \cosh z_5 C_{35} - \sinh z_5 C_{45} = 0 \dots (7)$$

$$\sin z_5 C_{15} + \cos z_5 C_{25} - \sinh z_5 C_{35} - \cosh z_5 C_{45} = 0 \dots (6)$$

$$C_{23} + C_{43} = 0 \quad \dots\dots\dots(17)$$

$$a_1(\cos z_1 C_{11} - \sin z_1 C_{21} + \cosh z_1 C_{31} + \sinh z_1 C_{41}) = a_2(C_{12} + C_{32}) \quad \dots\dots\dots(18)$$

$$a_2(C_{12} + C_{32}) = a_3(C_{13} + C_{33})$$

which could be written in the following form:

$$C_{12} + C_{32} - B_{23} C_{13} - B_{23} C_{33} = 0 \quad \dots\dots\dots(19)$$

$$E I_1 a_1^2 (-\sin z_1 C_{11} - \cos z_1 C_{21} + \sinh z_1 C_{31} + \cosh z_1 C_{41}) - E I_2 a_2^2 (-C_{22} + C_{42}) = E I_3 a_3^2 (-C_{23} + C_{43}) \quad \dots\dots\dots(20)$$

$$E I_2 a_1^3 (-\cos z_1 C_{11} + \sin z_1 C_{21} + \cosh z_1 C_{31} + \sinh z_1 C_{41}) - E I_2 a_2^3 (-C_{12} + C_{32}) = -M_1 w^2 (C_{22} + C_{42}) \quad \dots\dots\dots(21)$$

Equations (8), (10), (9), (11), (16), (20), (18) and (21) could be replaced by equations ( $\bar{8}$ ), ( $\bar{10}$ ) ..... by combining each two successive equations. This gives:-

$$\sin z_3 C_{13} + \cos z_3 C_{23} + \beta_{34} C_{24} + \alpha_{34} C_{34} = 0 \quad (\bar{8})$$

$$\sinh z_3 C_{33} + \cosh z_3 C_{43} + \alpha_{34} C_{24} + \beta_{34} C_{44} = 0 \quad (\bar{10})$$

$$\cos z_3 C_{13} - \sin z_3 C_{23} + \Delta_{34} C_{14} + \gamma_{34} C_{34} = 0 \quad (\bar{9})$$

$$\cosh z_3 C_{33} + \sinh z_3 C_{43} + \gamma_{34} C_{14} + \Delta_{34} C_{34} = 0 \quad (\bar{11})$$

$$\sin z_1 C_{11} + \cos z_1 C_{21} + \beta_{12} C_{22} + \alpha_{12} C_{42} - D_{12} D_{23} C_{23} = 0 \quad (\bar{16})$$

$$\sinh z_1 C_{31} + \cosh z_1 C_{41} + \alpha_{12} C_{22} + \beta_{12} C_{42} + D_{12} D_{23} C_{23} = 0 \quad (\bar{20})$$

$$\cos z_1 C_{11} - \sin z_1 C_{21} + \Delta_{12} C_{12} - R_1 z_1 C_{22} + \gamma_{12} C_{32} - R_1 z_1 C_{42} = 0 \quad (\bar{18})$$

$$\cosh z_1 C_{31} + \sinh z_1 C_{41} - \gamma_{12} C_{12} + R_1 z_1 C_{22} + \Delta_{12} C_{32} + R_1 z_1 C_{42} = 0 \quad (\bar{21})$$

where;

$$a_r = \frac{z_r}{L_r}$$

$$B_{r(r+1)} = \frac{a_{r+1}}{a_r}$$

$$D_{r(r+1)} = B_{r(r+1)}^2 \cdot \frac{I_{r+1}}{I_r}$$

$$\alpha_{r(r+1)} = 0.5 (D_{r(r+1)} - 1)$$

$$\beta_{r(r+1)} = -(\alpha_{r(r+1)} + 1) = -0.5(D_{r(r+1)} + 1)$$

$$\gamma_{r(r+1)} = \alpha_{r(r+1)} \cdot B_{r(r+1)}$$

$$\Delta_{r(r+1)} = \beta_{r(r+1)} \cdot B_{r(r+1)}$$

$$R_1 = \frac{M_1}{2 m_1}$$

Simplifying equation (15) by the following substitution:

$$U = \cos z_2 + R_2 z_2 \sin z_2$$

$$V = -\sin z_2 + R_2 z_2 \cos z_2$$

$$W = -\cosh z_2 + R_2 z_2 \sinh z_2$$

$$X = -\sinh z_2 + R_2 z_2 \cosh z_2$$

where

$$\frac{m_5 w^2}{E I_2 a_2^3} = \frac{m_5}{m_2} z_2^2 = R_2 z_2^2$$

gives

$$U C_{12} + V C_{22} + W C_{32} + X C_{42} = 0 \quad \dots\dots\dots(15)$$

Equations from (2) to (7), from (8) to (11), from (12) to (14), (15), (16), (17), (18), (19), (20) and (21) form a set of equations containing the unknown constants  $C_{11}, C_{12}, \dots, C_{45}$  and their unknown coefficients. Since the values of these constants are not zero, hence the set of equations yield to the frequency equation in the form of 20th order determinant, after reduction to 17th order and rearrangement the determinant equation (22) is established.

$$\begin{array}{r}
\sin z_1 \cos z_1 \\
\cos z_1 - \sin z_1 \\
\sinh z_1 \cosh z_1 \\
\cosh z_1 \sinh z_1 \\
\sin z_2 \cos z_2 \\
\cos z_2 - \sin z_2 \\
\sinh z_2 \cosh z_2 \\
\cosh z_2 \sinh z_2 \\
U \\
1
\end{array}
\begin{array}{r}
\beta_n \\
-\alpha_n \\
-\beta_n \\
-\alpha_n \\
\beta_n \\
-\alpha_n \\
-\beta_n \\
-\alpha_n \\
V \\
1
\end{array}
\begin{array}{r}
\alpha_n \\
-\beta_n \\
-\alpha_n \\
-\beta_n \\
\alpha_n \\
-\beta_n \\
-\alpha_n \\
-\beta_n \\
W \\
1
\end{array}
\begin{array}{r}
-D_{12}D_{23} \\
-D_{12}D_{23} \\
-2D_{25} \\
-B_{25} \\
-B_{23} \\
\sin z_3 \cos z_3 \\
\cos z_3 - \sin z_3 \\
-\cosh z_3 \sinh z_3 \\
-\sinh z_3 \cosh z_3 \\
-B_{23} \\
\sin z_3 \cos z_3 \\
\cos z_3 - \sin z_3 \\
-\cosh z_3 \sinh z_3 \\
-\sinh z_3 \cosh z_3 \\
-B_{23} \\
\sin z_3 \cos z_3 \\
\cos z_3 - \sin z_3 \\
-\cosh z_3 \sinh z_3 \\
-\sinh z_3 \cosh z_3 \\
X \\
1
\end{array}
\begin{array}{r}
\alpha_{34} \\
\beta_{34} \\
\alpha_{34} \\
\beta_{34} \\
\Delta_{34} \\
\beta_{34} \\
\Delta_{34} \\
\sin z_4 \cos z_4 - \sinh z_4 - \cosh z_4 \\
\cos z_4 - \sin z_4 - \cosh z_4 - \sinh z_4 \\
\sin z_4 \cos z_4 - \sinh z_4 - \cosh z_4 \\
\cos z_4 - \sin z_4 - \cosh z_4 - \sinh z_4 \\
\sin z_5 \\
\cos z_5 \\
-\sin z_5 \\
\cos z_5 + \sinh z_5 \\
\cosh z_5 \\
-\sinh z_5 \\
-\cosh z_5 \\
+\sinh z_5
\end{array}$$

The values of  $Z_2, Z_3, Z_4$  and  $Z_5$  could be determined in terms of  $Z_1$  from the equation (2):

$$Z_{r+1} = Z_r \cdot \frac{L_{r+1}}{L_r} B_{r(r+1)}.$$

Substituting these values into equation (22) yields the frequency equation in terms of  $Z_1$  alone.

#### 5 - THE NATURAL FREQUENCIES AND MODERFORMS OF THE CONSIDERED MACHINE:

The different natural frequencies of the machine could be determined from the following equation (2):

$$f = \frac{Z_1^2}{2 \times L_1^2} \sqrt{\frac{E I_1 g}{P A_1}} \quad \text{CPS}$$

by substituting the different values of ( $Z_1$ ). The corresponding modeforms of the machine could be determined by calculating the deflection of each beam using equation (1). For this purpose the values of  $C_{11}, C_{21}, \dots, C_{45}$  should be evaluated. This could be done by solving the set of equations given in section (4). This set of equations could be first reduced to nineteen equations by omitting the last equation and substituting  $C_{11} = 1$ . The moderform is only found on a relative basis, so that all the modeform is only found on a relative basis, so that all the modeform equation (1) can be multiplied on the right-hand side by the same arbitrary constant  $\neq 0$ .

#### 6 - THE BASIC MODEL OF THE MACHINE AND THE SUGGESTED MODIFICATIONS:

The basic model of the machine is shown in Fig. (2). Four modifications are suggested and shown in Figs (5), (6), (7) and (8). All modifications are done in such a way that the column of the machine has the same mass, height and wall thickness, whilst the dimensions and weights of the table and overarm are kept constant. In the first modification the column has vertical sides and tapered back, whilst in the second the column assumes squared cross-section. In the third and fourth modifications the back side wall is given a circular shape. The

idealized models of the four suggested modifications are shown in Figs. (9), (10), (11) and (12).

The configuration of the overarm of the machine is also investigated. Five different cross-sectional alternative shapes of the overarm are suggested and shown in Fig. (13). The five suggested overarms have the same mass and length.

The investigation of the dynamic behaviour of the basic and suggested models will be done in the following way:-

1. The natural frequencies and modeforms of the basic model will be first determined and considered as basic data.
2. The natural frequencies and modeforms of each of the suggested modifications will be measured and compared with the basic data in order to evaluate their effects. In this way the suggested model which will have the superior properties will be detected.
3. The dynamic behaviour of the superior model will be further investigated by changing its overarm in the sequence shown in Fig. (13).

#### 7 - RESULTS, DISCUSSION AND CONCLUSIONS:

The results of this investigation are tabulated and plotted in the form shown hereafter.

Table (1) shows the characteristics of the basic model of the machine, while tables (2) to (5) show the characteristics of each suggested modification. Tables (6) to (9) give the results of comparison between the characteristics of the suggested modifications and the basic model, which show that the third and fourth suggested modified models are superior than the basic.

The characteristics of the third modified model when equipped with the different suggested shapes of overarms, shown in Fig. (13), are included in tables (10) to (14). Tables (15) to (19) show the results of comparison between the characteristics of this model, when equipped with the different overarm shapes, with its basic model. The basic model in this case will be the third modified model equipped with the original overarm as it



is shown in Fig. (7). Hence the results given in table (4) will be the basic data. Tables (15) to (19) show that the third modification equipped with overarm possessing the second and/or the fifth shapes gives the best results. These two combinations will be termed (III - 2) and (III - 5).

Tables (20) to (24) are concerned with the characteristics of the fourth modification when equipped with the different suggested shapes of overarms, whilst tables (25) to (29) give the results of comparison between its characteristics and that of its basic model. The basic model in this case will be the fourth modification equipped with the original overarm as it is shown in Fig. (8), and its data given in table (5) will be considered as basic data. The results of comparison show that the fourth modification equipped with overarm possessing the second and/or the fifth shapes gives the best results. These two combinations will be termed (IV - 2) and (IV - 5).

The main results of this investigation are summarized in Figs (14) to (18). These figures compare the characteristics of the basic model of the machine with those of the four best models, namely, (III - 2), (III - 5), (IV - 2) and (IV - 5). Regarding the moderforms, the best four models behave approximately in the same manner, as it is shown in Fig. (14), and from Figs (15) to (17) which show the deflections of the main parts of each model at the different modes. However, the modified model (III - 2) may be the best one, in the sense that, it ensures, the minimum deflection and hence the maximum rigidity. The percentage reduction in the deflection of the main parts of that model is given in Fig. (18), which shows how far the deflection of each part of the suggested model is reduced in comparison with those of the basic model.

The percentage reduction reaches 21% for the column, 23% for the table and 71% for the overarm. Further investigation on prototype scale seems to be necessary in order to indentify securely the suggested model which could be recommended.

Regarding the magnitude of the natural frequencies, it could be seen that, the first natural frequencies of the best model

range from 203 to 221 C/S compared to 181 C/S for the basic model of the machine. This gives a mean increase of about 31 C/S corresponding to 1860 r.p.m., which makes the first natural frequencies of the suggested models far away, with an example space, from the working range of the machine.

In order to visualize the individual effects of the column and hence the overarm, the set of results illustrated in Figs (19) to (22) is presented. Figs (19) to (20) compare the deflections of the different main parts of the basic, third and fourth suggested models when equipped with the original overarm. On the other hand, Fig. (22) indicates the percentage reduction reaches 4% for the column, 9% for the table and 28% for the overarm.

From the presented results and discussion the following main conclusions could be drawn:-

1. When the model mass and main dimensions are kept constant, the rigidity of a machine tool structure could be increased by altering its configuration. The optimum configuration could be detected by checking the various alternative possible shapes of its main parts.
2. For the purpose of checking the different possible configurations of a machine tool structure, the classical beam theory may be one of the useful tools.
3. The four suggested models for the considered milling machine exhibit reduced deformation and increased natural frequency.
4. The percentage reduction in deformation of the suggested models amounts to 21% for the column, 23% for the table and 71% for the overarm. The natural frequency increased by about 31 C/S.
5. Rounding the back-side of the column and/or the overarm yield considerable improvement of the dynamic characteristics of the milling machine.
6. Rounding the back-side of the column of the horizontal milling machine while keeping the other parts having the original shapes results in reducing the deflection by about 4% for the column, 9% for the table and 28% for the overarm.

NOMENCLATURE

$A_r$	Cross-sectional area.
$a_r$	Frequency constant.
$B_{r(r+1)}$	Area constant.
$C_{1r}, C_{2r}, \dots, C_{4r}$	Boundary constants.
$E$	Modulus of elasticity.
$E I$	Flexural rigidity.
$f$	Frequency.
$g$	Acceleration of gravity.
$I_r$	Moment of inertia of area.
$L_r$	Length.
$m_r$	Mass.
$R_1, R_2$	Masses ratio.
$r$	Subscript taking the values 1, 2, ..., 5 and indicates the number of beam.
$w$	Circular frequency.
$X_r$	Abscissa.
$Y_r$	Ordinate.
$Z_r$	frequency constant.
	Density of beam material.
$\rho A$	Mass per unit length.

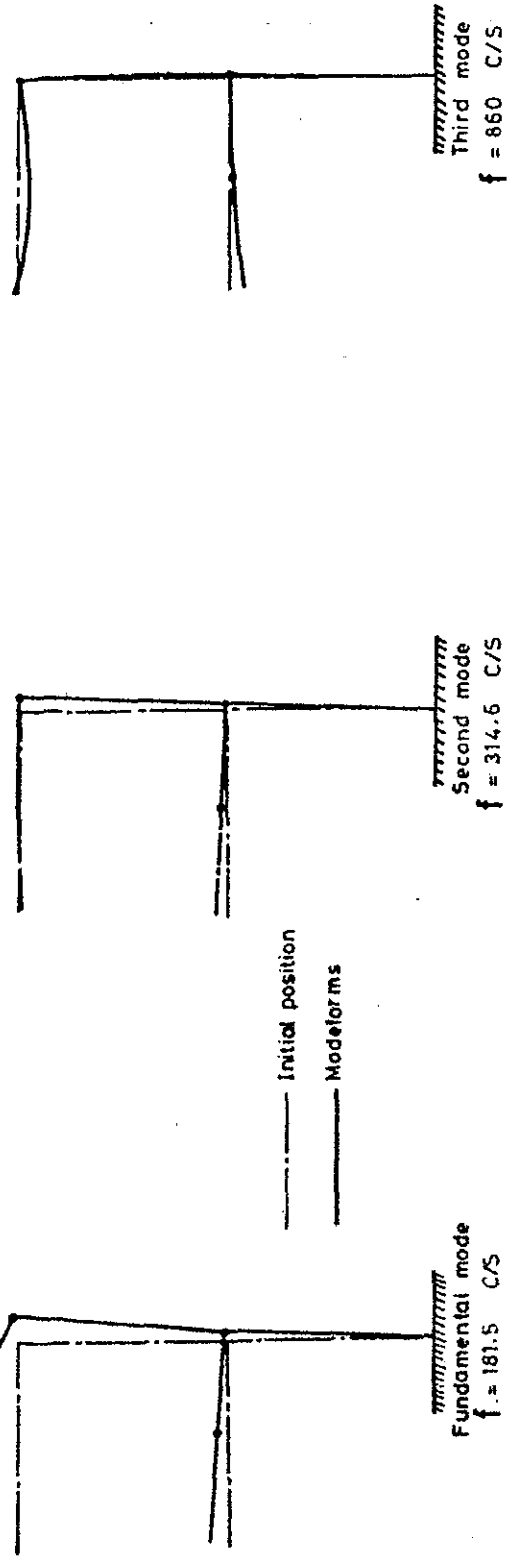
REFERENCES

1. S. Taylor and S.A. Tobias, "Lumped - constants method for the prediction of the vibration characteristics of machine tool structures". Proc. 5th Inter. M.T.D.R. Conference, 1964, pp. 37-52.
2. J.C. Maltbaek, "Classical beam method for the prediction of vibration characteristics of machine structures", Proc. 5th Inter. M.T.D.R. Conference, 1964, pp. 23-35.
3. A. Cowley, "Analysis of machine tool structure by computing techniques", Proc. Inter. M.T.D.R. Conference, 1967, pp. 119-137.
4. S. Taylor, "A computer analysis of an openside planing machine", Proc. Inter. M.T.D.R. Conference, 1965, pp. 197-206.
5. S. Taylor, "The design of machine tool structures using a digital computer", Proc. Inter. M.T.D.R. Conference, 1966, pp. 369-383.
6. S. Taylor, "Computer aided design of a planing machine structure", Proc. Inter. M.T.D.R. Conference, 1968, pp. 739-750.

7. A.C. Stephen and S. Taylor, "Computer analysis of machine tool structures by the finite element method", Proc. Inter. M.T.D.R. Conference, 1968, pp. 751-761.
8. H. Optiz and R. Noppen, "Evaluation of computer aided methods for the design of machine tool structures", Ann. CIRP, Vol. 2212, 1973, pp. 227-231.
9. A. El Hakim and Z. Magdedurg, "Application of classical beam theory of investigation of natural frequencies Maschinenbautechnik, 1970, pp. 537-539.
10. S. Timoshenko, "Vibration problems in engineering", 3rd ed, Van Nostrand, New York (1955).

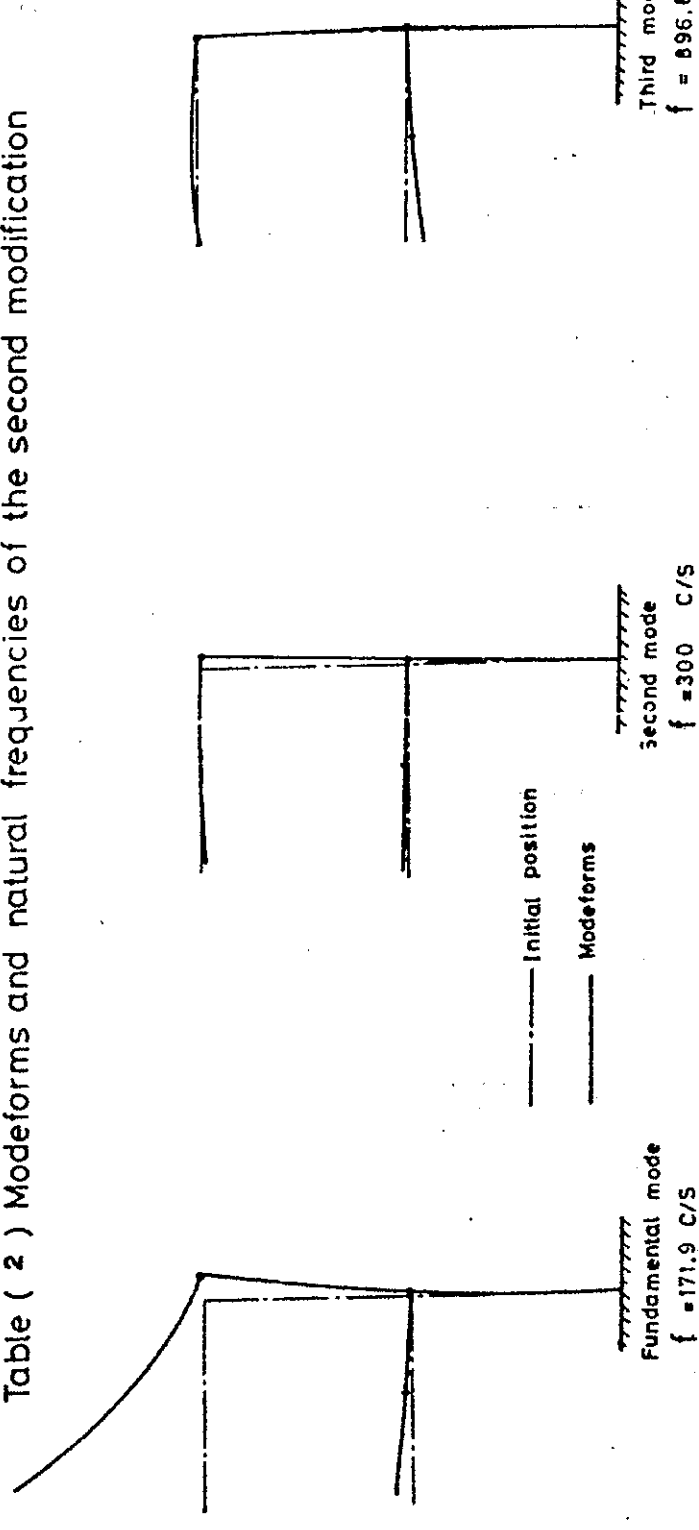
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		Beam 1			Beam 2			Beam 3			Beam 4			Beam 5												
		$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$												
$z_1$	$f$	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1										
1 <sup>st</sup>	0.79	181.5	0	0.09	0.34	0.74	1.28	1.93	2.68	3.35	4.48	0	0.3	0.61	0.92	1.23	1.73	1.54	1.87	2.18	2.5	0	4.51	14.22	26.67	39.98
2 <sup>nd</sup>	1.04	314.6	0	0.08	0.28	0.59	0.96	1.34	1.72	2.08	2.42	0	0.2	0.41	0.63	0.85	1.08	1.31	1.55	1.79	0	0	0.18	0.09	-0.17	-0.51
3 <sup>rd</sup>	1.72	860.5	0	0.11	0.35	0.56	0.59	0.51	0.41	0.28	0.12	0	-0.1	-0.31	-0.59	-0.94	-1.34	-1.8	-2.29	-2.79	0	-1.02	-2.08	-1.56	0.38	

Table ( 1 ) Modeforms and natural frequencies of the basic model.



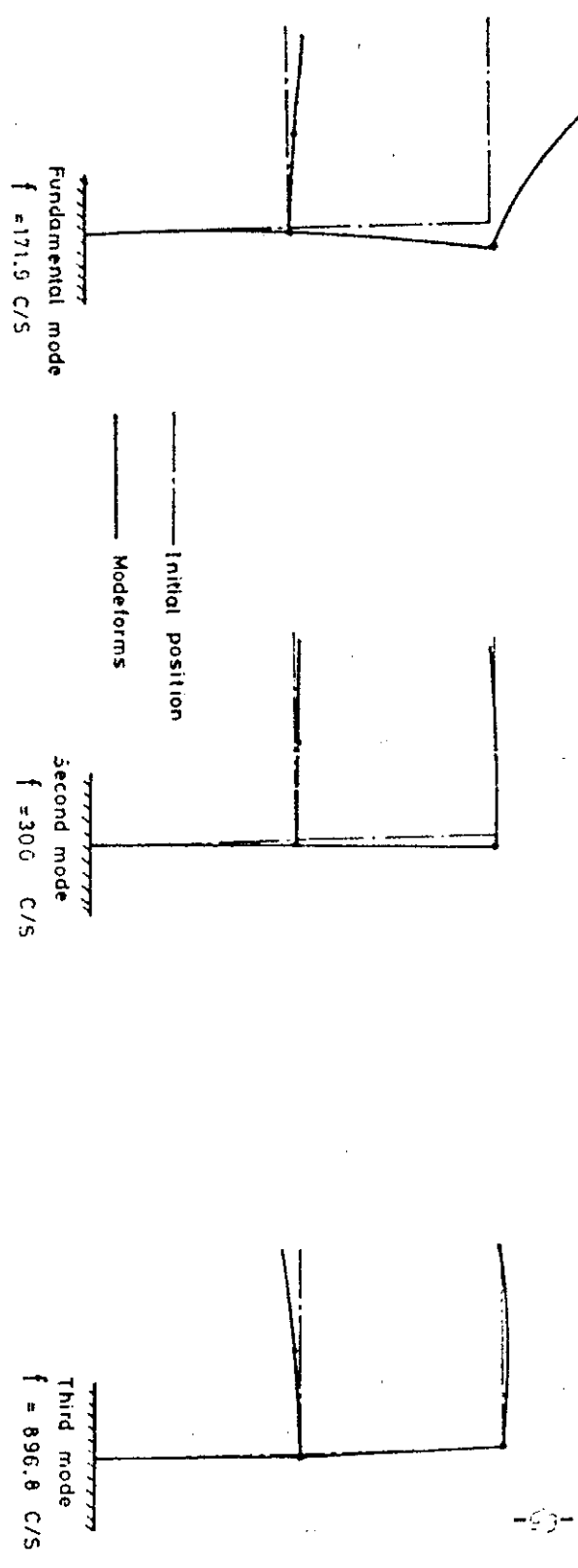
		Deflections																									
Mode no.	Z <sub>1</sub>	f c/s	Beam 1			Beam 2			Beam 3			Beam 4			Beam 5												
			X <sub>1</sub> /L <sub>1</sub>			X <sub>2</sub> /L <sub>2</sub>			X <sub>3</sub> /L <sub>3</sub>			X <sub>4</sub> /L <sub>4</sub>			X <sub>5</sub> /L <sub>5</sub>												
			0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1					
1 <sup>st</sup>	0.81	171.9	0	0.09	0.35	0.76	1.31	1.31	1.98	2.75	3.62	4.59	0	0.3	0.6	0.91	1.21	1.21	1.52	1.83	2.14	2.46	0	4.19	13.03	24.27	36.25
2 <sup>nd</sup>	1.07	300	0	0.08	0.29	0.6	0.97	0.97	1.36	1.73	2.08	2.41	0	0.19	0.39	0.59	0.8	0.8	1.02	1.23	1.45	1.67	0	0.16	0.04	-0.25	-0.36
3 <sup>rd</sup>	1.85	896.8	0	0.13	0.4	0.61	0.59	0.59	0.44	0.33	0.25	0.19	0	-0.15	-0.41	-0.76	-1.17	-1.17	-1.65	-2.2	-2.76	-3.36	0	0.58	1.24	0.84	-0.51

Table ( 2 ) Modeforms and natural frequencies of the second modification



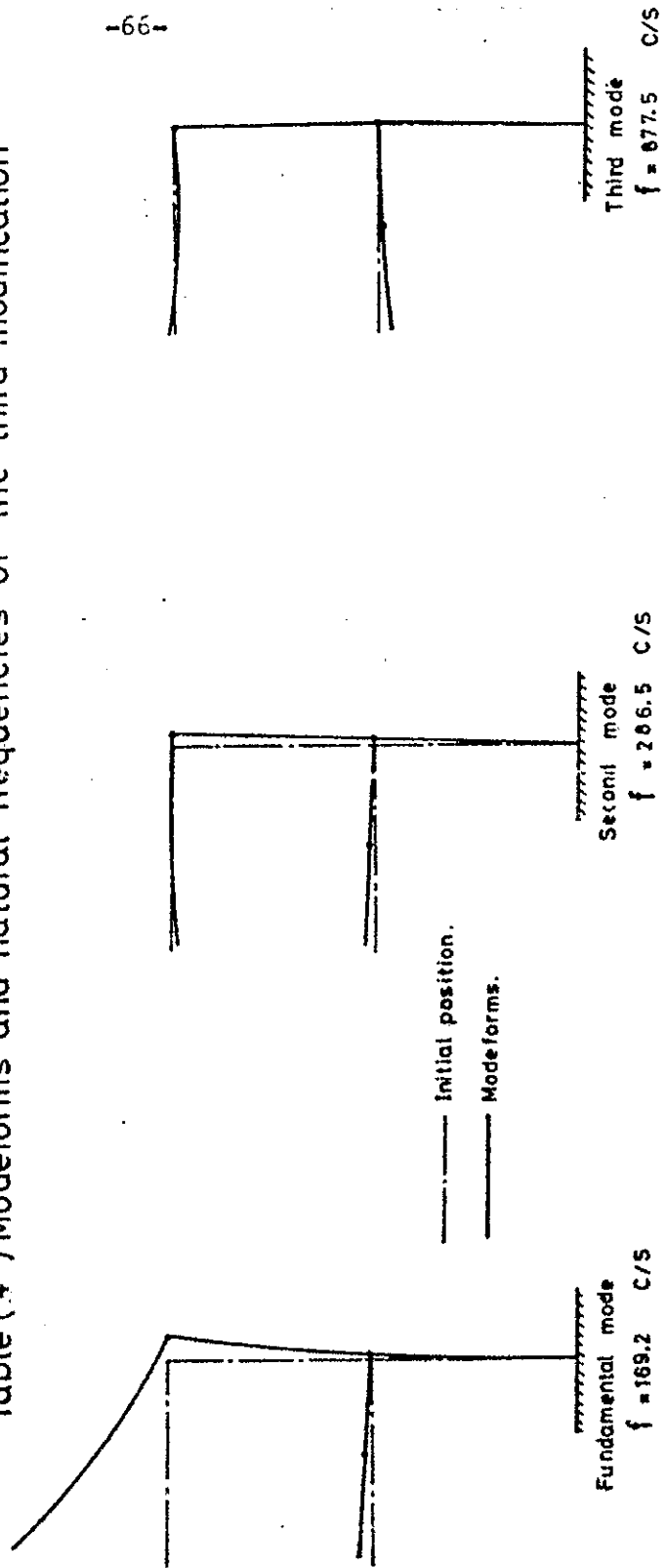
Mode no.	z <sub>1</sub>	f c/s	Deflections																									
			Beam 1			Beam 2			Beam 3			Beam 4			Beam 5													
			X <sub>1</sub> /L <sub>1</sub>	X <sub>2</sub> /L <sub>2</sub>	X <sub>3</sub> /L <sub>3</sub>	X <sub>4</sub> /L <sub>4</sub>	X <sub>5</sub> /L <sub>5</sub>																					
1 <sup>st</sup>	0.81	171.9	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1						
2 <sup>nd</sup>	1.07	300	0	0.08	0.29	0.6	0.97	0	0.97	1.36	1.73	2.08	2.41	0	0.19	0.39	0.59	0.8	0.8	1.02	1.23	1.45	1.67	0	0.16	0.04	-0.25	-0.36
3 <sup>rd</sup>	1.85	896.8	0	0.13	0.4	0.61	0.59	0	0.59	0.44	0.33	0.25	0.19	0	-0.15	-0.41	-0.76	-1.17	-1.17	-1.65	-2.2	-2.76	-3.36	0	0.58	1.24	0.84	-0.51

Table ( 3 ) Modeforms and natural frequencies of the second modification



Mode no.	z <sub>1</sub>	f c/s	Deflections																							
			Beam 1			Beam 2			Beam 3			Beam 4			Beam 5											
			X <sub>1</sub> /L <sub>1</sub>			X <sub>2</sub> /L <sub>2</sub>			X <sub>3</sub> /L <sub>3</sub>			X <sub>4</sub> /L <sub>4</sub>			X <sub>5</sub> /L <sub>5</sub>											
1 <sup>st</sup>	0.83	169.2	0	0.09	0.34	0.74	1.27	1.91	2.64	3.46	4.37	0	0.28	0.56	0.84	1.12	1.12	1.41	1.7	1.99	2.28	0	3.46	10.62	19.7	29.4
2 <sup>nd</sup>	1.06	266.5	0	0.08	0.29	0.59	0.96	1.33	1.68	2.01	2.31	0	0.16	0.37	0.56	0.75	0.75	0.95	1.15	1.35	1.55	0	0.03	-0.34	-0.93	-1.57
3 <sup>rd</sup>	1.89	572.5	0	0.14	0.41	0.61	0.57	0.37	0.2	0.04	-0.13	0	-0.16	-0.41	-0.73	-1.11	-1.11	-1.55	-2.04	-2.56	-3.06	0	-0.39	-0.66	-0.33	0.45

Table (4) Modeforms and natural frequencies of the third modification



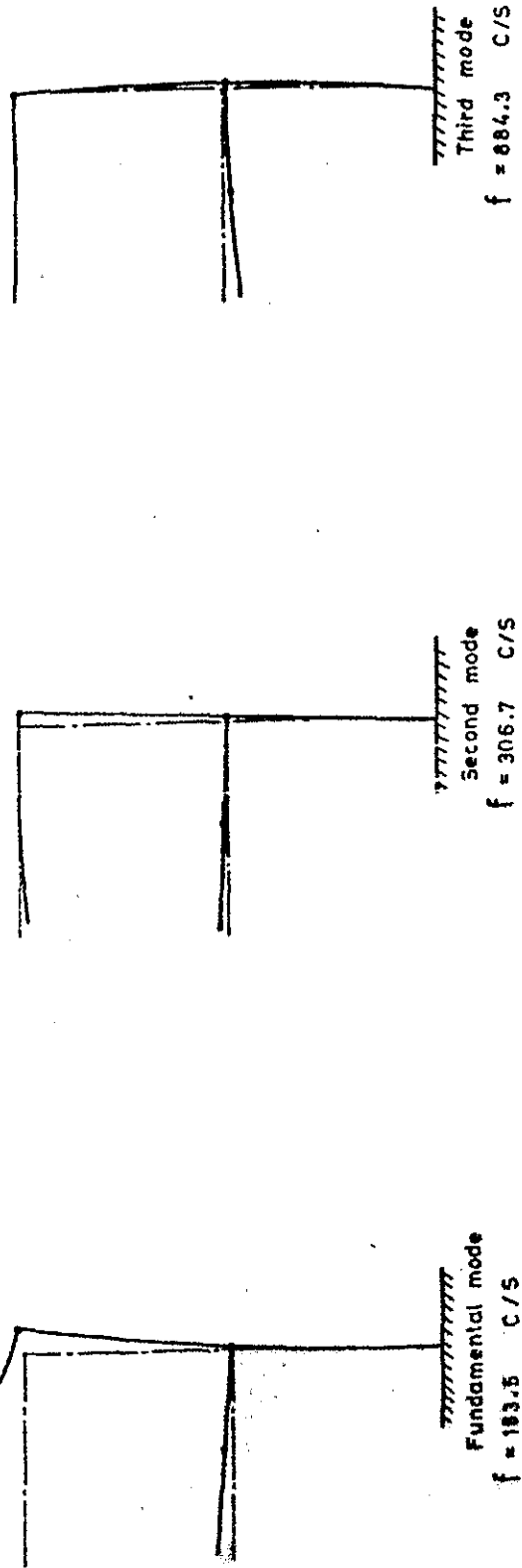


Mode no.	z	f C/S	Deflections																							
			Beam 1			Beam 2			Beam 3			Beam 4			Beam 5											
			$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$											
1st	0.82	182.5	0	0.08	0.33	0.73	1.25	1.75	1.87	2.59	3.4	4.3	0	0.28	0.57	0.86	1.15	1.42	1.74	2.03	2.33	0	3.77	11.75	21.89	32.7
2nd	1.06	306.7	0	0.08	0.29	0.59	0.95	1.33	1.69	2.03	2.35	0	0.19	0.38	0.59	0.79	0.79	1	1.22	1.43	1.65	0	0.05	-0.27	-0.81	-1.41
3rd	1.6	884.3	0	0.12	0.38	0.58	0.59	0.59	0.46	0.35	0.25	0.15	0	-0.13	-0.37	-0.69	-1.07	-1.52	-2.02	-2.55	-3.1	0	-0.04	0.01	0.09	0.15

Table ( 5 ) Modeforms and natural frequencies of the fourth modification

----- Initial position.

----- Modeforms.



Mode number	Deflection ratios																			
	Beam 1			Beam 2			Beam 3			Beam 4			Beam 5							
	$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$							
0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	
1 <sup>st</sup> mode	0	1	1.03	1.02	1.02	1.01	1.01	1.01	1.01	0	0.93	0.93	0.93	0.94	0.94	0.94	0.94	0.95	0.93	0
2 <sup>nd</sup> mode	0	1	1.03	1.01	1.01	1	0.98	0.97	0.95	0	0.9	0.89	0.88	0.88	0.88	0.88	0.88	0.87	0.86	0
3 <sup>rd</sup> mode	0	1.27	1.2	1.12	0.96	0.68	0.46	0.21	-0.25	0	1.7	1.29	1.22	1.22	1.22	1.18	1.15	1.13	1.11	0

Table ( 6 ) Comparison between the first modification and basic model .

Mode number	Deflection ratios																			
	Beam 1			Beam 2			Beam 3			Beam 4			Beam 5							
	$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$							
0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	
1 <sup>st</sup> mode	0	1	1.03	1.02	1.02	1.02	1.02	1.02	1.02	0	1	0.98	0.98	0.98	0.98	0.98	0.99	0.98	0.98	0
2 <sup>nd</sup> mode	0	1	1.03	1.01	1.01	1.01	1.01	1	0.99	0	0.95	0.95	1.07	0.94	0.94	1.06	0.93	0.93	0.93	0
3 <sup>rd</sup> mode	0	1.18	1.14	1.09	1	1	0.86	0.8	0.89	1.58	0	1.5	1.32	1.29	1.24	1.24	1.23	1.22	1.21	0

Table( 7 ) Comparison between the second modification and basic model .

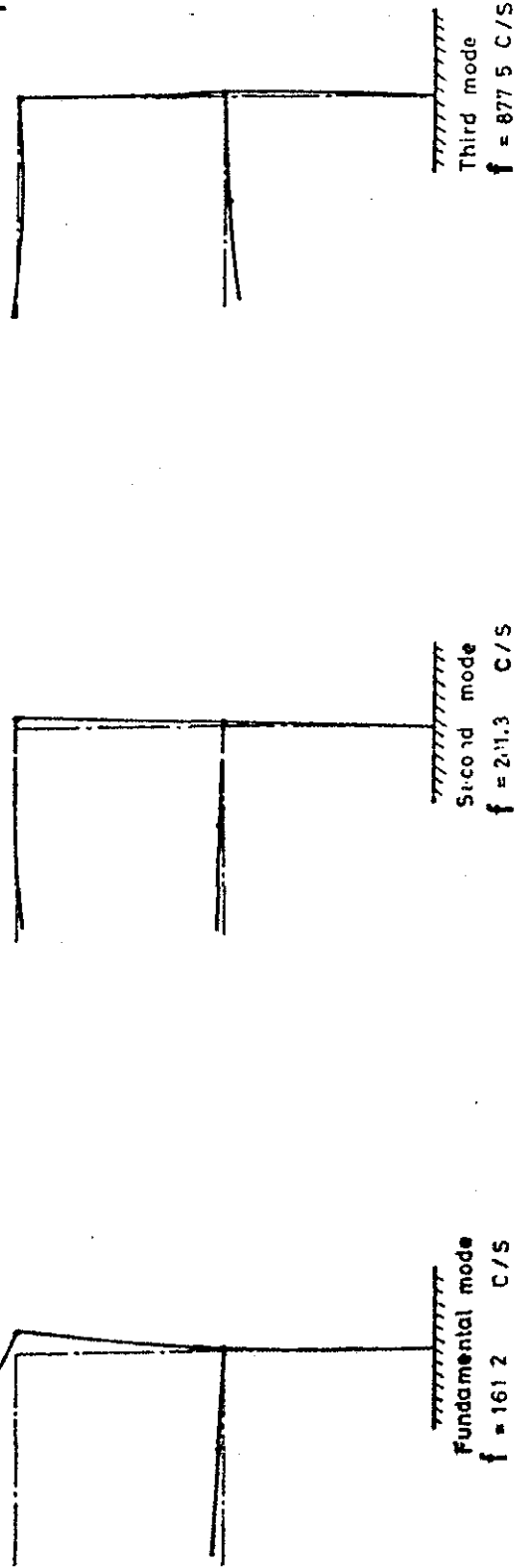


Mode no.	$z_1$	f C/S	Deflections																								
			Beam 1			Beam 2			Beam 3			Beam 4			Beam 5												
			$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$												
1 <sup>st</sup>	0.81	161.2	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1					
2 <sup>nd</sup>	1.07	281.3	0	0.09	0.35	0.75	1.3	1.3	1.96	2.73	3.59	4.55	0	0.28	0.57	0.87	1.16	1.16	1.45	1.75	2.05	2.34	0	4.16	12.96	24.16	36.11
3 <sup>rd</sup>	1.89	877.5	0	0.02	0.29	0.6	0.97	0.97	1.35	1.72	2.06	2.39	0	0.18	0.37	0.57	0.76	0.76	0.96	1.17	1.37	1.58	0	0.15	0.02	-0.28	-0.67
			0	0.14	0.41	0.61	0.57	0.57	0.37	0.2	0.04	0.13	0	-0.16	-0.41	-0.73	-1.12	-1.12	-1.55	-2.44	-2.56	-3.02	0	-0.43	-0.63	-0.27	0.65

Table ( 10 ) the third modification equipped with the first overarm

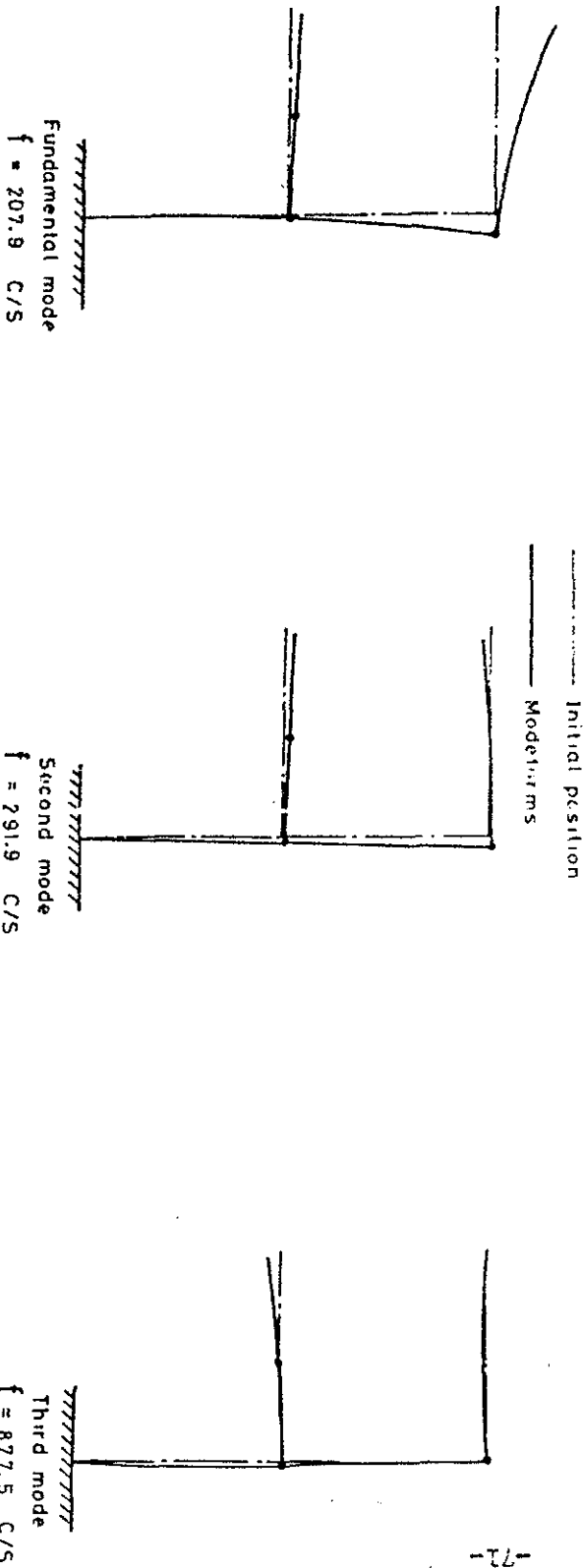
----- Initial position

----- Modeforms



Mode no.	$z_1$	$f$ C/S	Deflections																								
			Beam 1			Beam 2			Beam 3			Beam 4			Beam 5												
			$X_1/L_1$	$X_2/L_2$	$X_3/L_3$	$X_4/L_4$	$X_5/L_5$																				
1 <sup>st</sup>	0.29	207.9	0	0.08	0.31	0.66	1.11	1.11	1.64	2.23	2.68	3.55	0	0.73	0.47	0.71	0.95	0.95	1.19	1.44	1.69	1.93	0	1.56	4.38	7.84	11.51
2 <sup>nd</sup>	1.09	291.9	0	0.08	0.78	0.58	0.92	0.97	1.28	1.61	1.91	2.17	0	0.7	1.35	0.53	0.71	0.71	0.9	1.09	1.29	1.48	0	-0.05	-0.55	-1.29	-2.17
3 <sup>rd</sup>	1.89	877.5	0	0.14	0.41	0.61	0.57	0.57	0.37	0.2	0.03	-0.14	0	-0.16	0.41	-0.74	-1.12	-1.12	-1.56	-2.05	-2.57	-3.07	0	-0.32	-0.51	-0.53	-0.13

Table ( 11) The third modification equipped with the second overarm

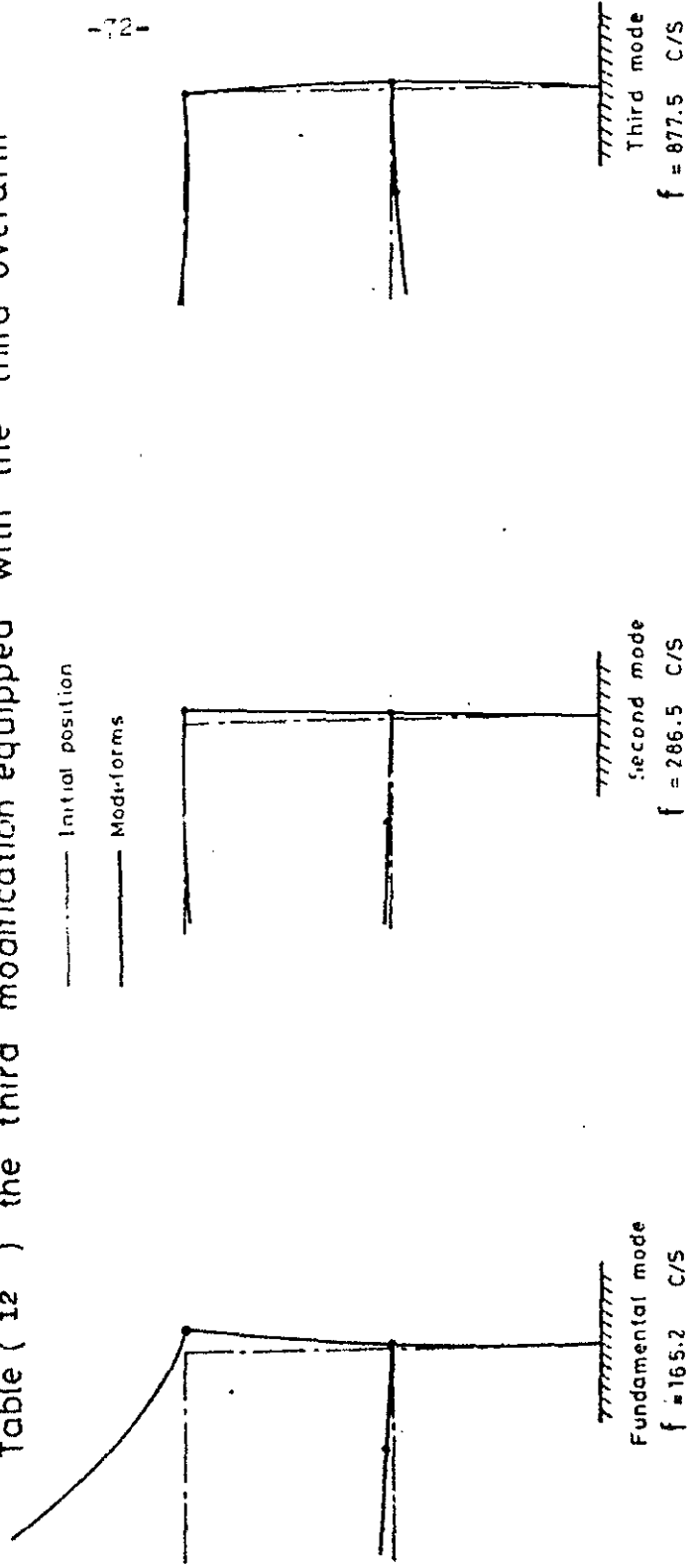


Mode no.		Deflections																								
		Beam 1				Beam 2				Beam 3				Beam 4				Beam 5								
		$X_1/L_1$				$X_2/L_2$				$X_3/L_3$				$X_4/L_4$				$X_5/L_5$								
	$z_1$	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1
1 <sup>st</sup>	0.82	0	0.09	0.34	0.75	1.29	1.29	1.94	2.69	3.53	4.47	0	0.28	0.57	0.86	1.15	1.15	1.44	1.73	2.03	2.37	0	3.87	11.97	22.27	33.25
2 <sup>nd</sup>	1.08	0	0.08	0.29	0.6	0.96	0.96	1.33	1.64	2.02	2.32	0	0.18	0.37	0.56	0.75	0.75	0.99	1.15	1.35	1.56	0	0.02	-0.37	-0.98	-1.64
3 <sup>rd</sup>	1.89	0	0.14	0.41	0.61	0.57	0.57	0.37	0.2	0.04	-0.13	0	-1.11	-0.41	-0.73	-1.11	-1.11	-1.55	-2.04	-2.56	-3.12	0	-0.41	-0.68	-0.29	0.57

Table ( 12 ) the third modification equipped with the third overarm

— Initial position

— Mod+forms

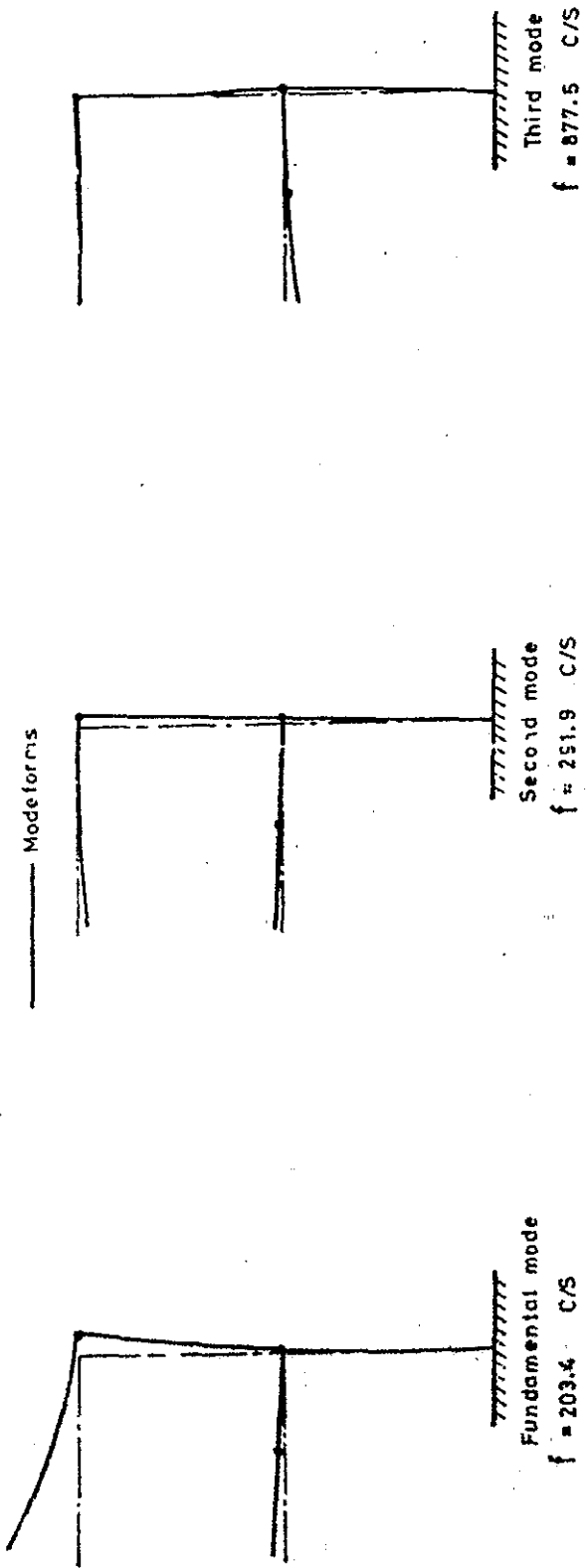




Mode no.	z <sub>1</sub> f c/s	Deflections																							
		Beam 1			Beam 2			Beam 3			Beam 4			Beam 5											
		X <sub>1</sub> /L <sub>1</sub>			X <sub>2</sub> /L <sub>2</sub>			X <sub>3</sub> /L <sub>3</sub>			X <sub>4</sub> /L <sub>4</sub>			X <sub>5</sub> /L <sub>5</sub>											
	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1					
1 <sup>st</sup>	0.91	0.08	0.31	0.67	1.13	1.13	1.67	2.27	2.93	3.64	0	0.27	0.48	0.72	0.97	0.97	1.22	1.47	1.72	1.97	0	1.69	4.79	8.63	12.69
2 <sup>nd</sup>	1.09	0.08	0.28	0.58	0.93	0.93	1.29	1.62	1.91	2.18	0	0.17	0.35	0.53	0.72	0.72	0.91	1.1	1.29	1.49	0	-0.04	-0.54	-1.28	-2.1
3 <sup>rd</sup>	1.89	0.14	0.41	0.61	0.57	0.57	0.37	0.2	0.03	-0.14	0	-0.13	-0.41	-0.74	-1.12	-1.12	-1.56	-2.05	-2.56	-3.09	0	-0.33	-0.61	-0.52	-0.1

Table ( 14 ) the third modification equipped with the fifth overarm

----- Initial position.  
 \_\_\_\_\_ Modeforms





Mode number	Frequency ratios.	Deflection ratios																							
		Beam 1			Beam 2			Beam 3			Beam 4			Beam 5											
		$X_1/L_1$	$X_2/L_2$	$X_3/L_3$	$X_4/L_4$	$X_5/L_5$	$X_1/L_1$	$X_2/L_2$	$X_3/L_3$	$X_4/L_4$	$X_5/L_5$	$X_1/L_1$	$X_2/L_2$	$X_3/L_3$	$X_4/L_4$	$X_5/L_5$									
1 <sup>st</sup> mode	0.95	0	1	1.03	1.01	1.02	1.02	1.02	1.03	1.04	1.04	0	1	1.02	1.03	1.03	1.03	1.03	1.02	0	1.2	1.22	1.22	1.23	
2 <sup>nd</sup> mode	0.98	0	1	1	1.01	1.01	1.01	1.01	1.02	1.02	1.03	0	1	1	1.02	1.01	1.01	1.02	1.01	1.02	0	5	-0.06	0.3	0.4
3 <sup>rd</sup> mode	1	0	1	1	1	1	1	1	1	1	1	0	1	1	1	1	1	1	1	1	0	1.1	1.04	0.82	1.44

Table ( 15 ) Comparison between the third modification equipped with the first overarm with its basic model.

Mode number	Frequency ratios.	Deflection ratios																								
		Beam 1			Beam 2			Beam 3			Beam 4			Beam 5												
		$X_1/L_1$	$X_2/L_2$	$X_3/L_3$	$X_4/L_4$	$X_5/L_5$	$X_1/L_1$	$X_2/L_2$	$X_3/L_3$	$X_4/L_4$	$X_5/L_5$	$X_1/L_1$	$X_2/L_2$	$X_3/L_3$	$X_4/L_4$	$X_5/L_5$										
1 <sup>st</sup> mode	1.23	0	0.89	0.91	0.89	0.87	0.87	0.86	0.84	0.82	0.81	0	0.82	0.84	0.84	0.85	0.85	0.84	0.85	0.85	0.84	0	0.45	0.41	0.4	0.34
2 <sup>nd</sup> mode	1.02	0	1	0.96	0.98	0.96	0.96	0.96	0.96	0.95	0.94	0	0.94	0.94	0.94	0.94	0.94	0.95	0.95	0.95	0.95	0	-1.66	0.15	1.39	1.35
3 <sup>rd</sup> mode	1	0	1	1	1	1	1	1	1	1.07	1.07	0	1	1	1.01	1	1	1	1	1	0	0.82	0.82	1.6	-0.29	

Table ( 16 ) Comparison between the third modification equipped with the second overarm with its basic model.



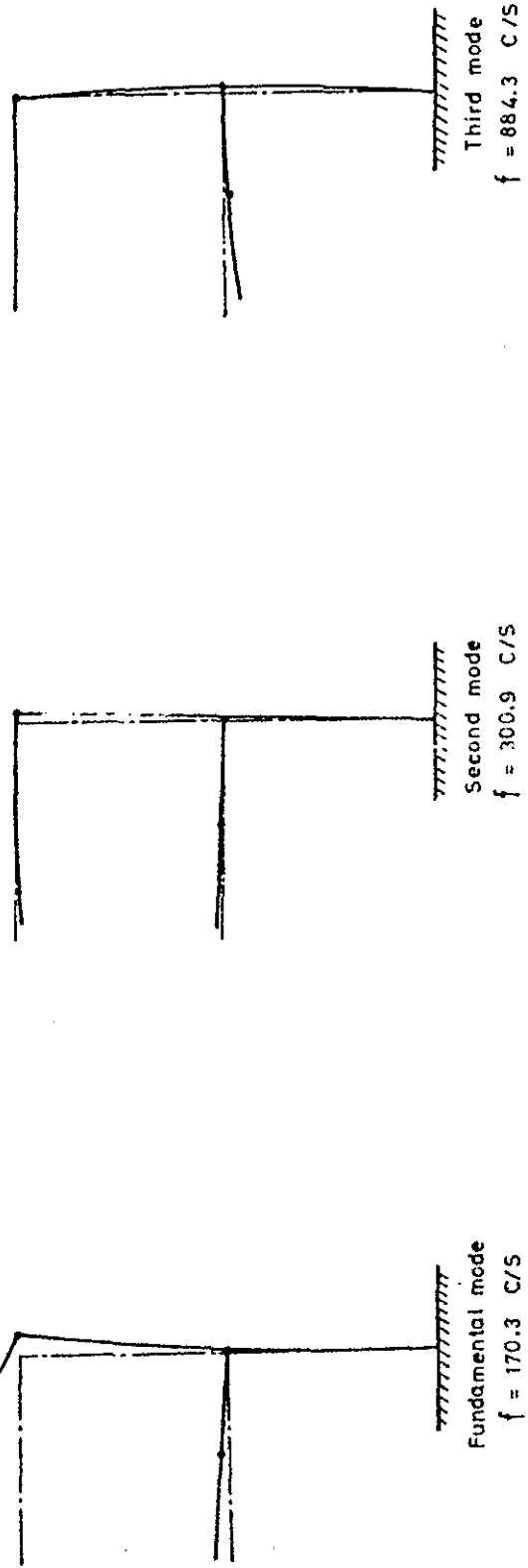
Mode number	Frequency ratios.	Deflections ratios																								
		Beam 1					Beam 2					Beam 3					Beam 4					Beam 5				
		$X_1/L_1$					$X_2/L_2$					$X_3/L_3$					$X_4/L_4$					$X_5/L_5$				
1 <sup>st</sup> mode	1.2	0	0.89	0.91	0.9	0.89	0.89	0.87	0.86	0.84	0.83	0	0.96	0.85	0.86	0.86	0.86	0.86	0.86	0	0.49	0.45	0.44	0.43		
2 <sup>nd</sup> mode	1.02	0	1	0.96	0.98	0.97	0.97	0.97	0.96	0.95	0.94	0	0.94	0.94	0.94	0.96	0.96	0.96	0.96	0	-1.33	1.59	1.37	1.34		
3 <sup>rd</sup> mode	1	0	1	1	1	1	1	1	1	1.07	0	1	1	1.01	1	1	1	1	1	0	0.84	0.92	1.57	0.22		

Table ( 19 ) Comparison between the third modification equipped with the fifth overarm with its basic model.

Mode no.	f c/s	Deflections																								
		Beam 1				Beam 2				Beam 3				Beam 4				Beam 5								
		$X_1/L_1$				$X_2/L_2$				$X_3/L_3$				$X_4/L_4$				$X_5/L_5$								
1st	170.3	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1
		0.09	0.34	0.75	1.29	1.29	1.29	1.69	2.72	3.59	4.57	0	0.23	0.59	0.9	1.2	1.2	1.51	1.82	2.12	2.43	0	4.74	14.99	28.11	42.14
2nd	300.9	0	0.08	0.29	0.59	0.96	1.35	1.72	2.08	2.43	0	0.13	0.39	0.6	0.81	0.81	1.02	1.24	1.46	1.68	0	0.2	0.17	0.04	-0.33	
3rd	884.3	0	0.12	0.38	0.56	0.59	0.46	0.35	0.25	0.15	0	-0.12	-0.37	-0.69	-1.07	-1.07	-1.52	-2.02	-2.56	-3.1	0	-0.03	0.02	0.09	0.13	

Table ( 20 ) the fourth modification equipped with the first overarm

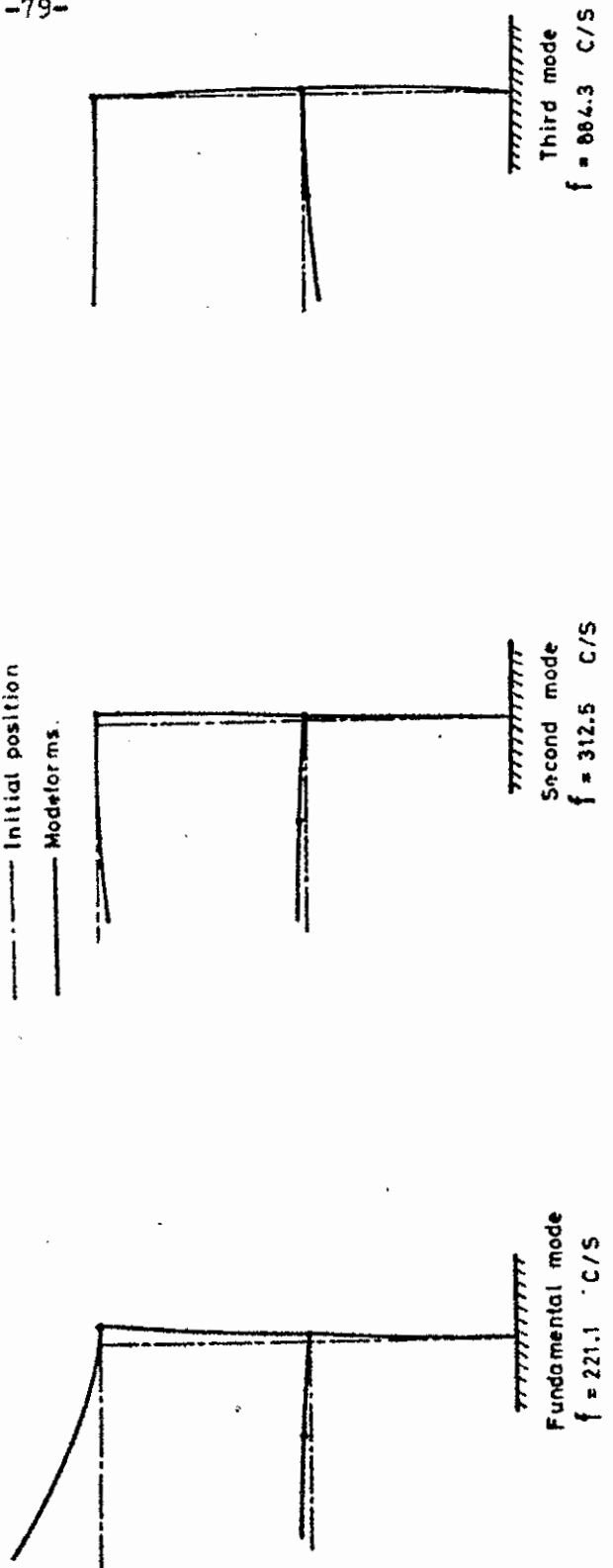
----- Initial position  
 \_\_\_\_\_ Modeforms



Mode no.	Z <sub>1</sub>	Deflections																								
		Beam 1					Beam 2					Beam 3					Beam 4					Beam 5				
		X <sub>1</sub> /L <sub>1</sub>					X <sub>2</sub> /L <sub>2</sub>					X <sub>3</sub> /L <sub>3</sub>					X <sub>4</sub> /L <sub>4</sub>					X <sub>5</sub> /L <sub>5</sub>				
	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	
1st	0.9	0.08	0.3	0.65	1.1	1.1	1.64	2.23	2.87	3.57	0	0.24	0.49	0.74	0.99	0.99	0.89	1.24	1.5	1.76	2.02	0	1.73	4.99	9.04	13.34
2nd	1.07	0.07	0.28	0.57	0.92	0.92	1.28	1.62	1.93	2.2	0	0.15	0.37	0.56	0.76	0.76	0.76	0.96	1.16	1.37	1.55	0	-0.04	-0.55	-1.31	-2.15
3rd	1.8	0.12	0.38	0.58	0.58	0.58	0.45	0.35	0.25	0.15	0	-0.13	-0.37	-0.7	-1.08	-1.08	-1.08	-1.45	-2.05	-2.59	-3.13	0	-0.02	0.05	0.14	0.2

Table (.21) the fourth modification equipped with the second overarm

----- Initial position  
 \_\_\_\_\_ Modeforms.

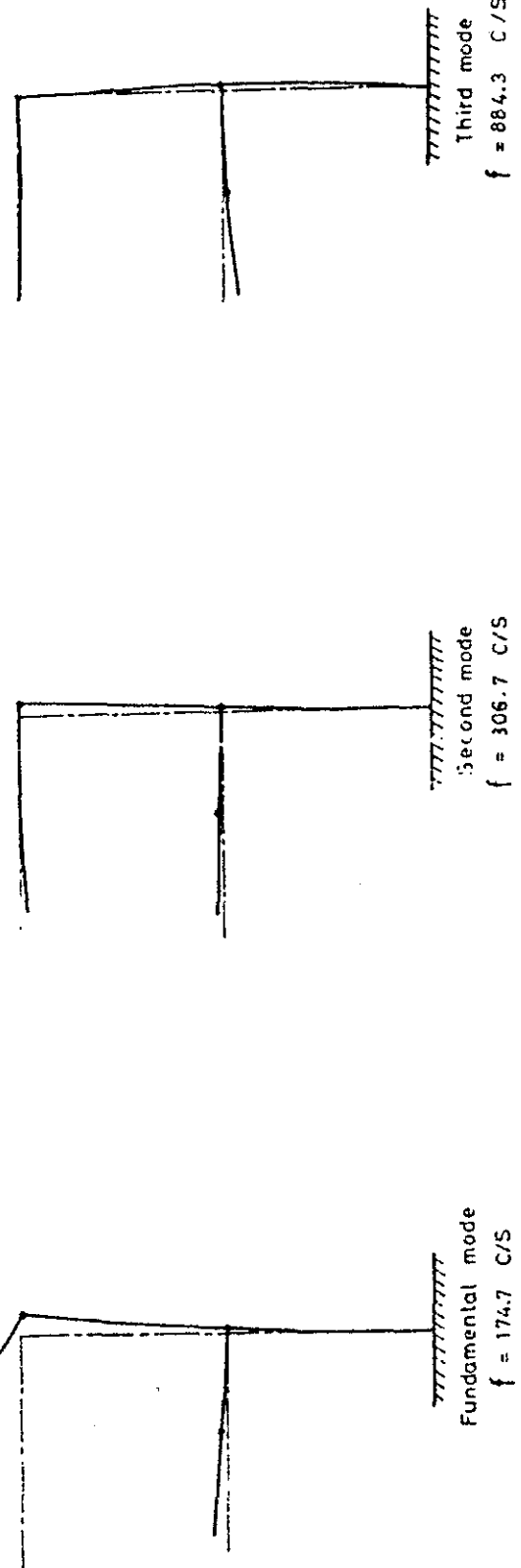


Mode no.	f c/s	Deflections																							
		Beam 1			Beam 2			Beam 3			Beam 4			Beam 5											
		$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$											
1 <sup>st</sup>	0.8	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1					
		0.05	0.34	0.74	1.28	1.28	1.93	2.69	3.53	4.48	0	0.29	0.59	0.89	1.19	1.19	1.49	1.79	2.1	2.41	0	4.3y	13.82	25.85	38.75
2 <sup>nd</sup>	1.06	0.08	0.25	0.59	0.95	0.95	1.33	1.69	2.04	2.36	0	0.19	0.38	0.59	0.76	0.76	1.01	1.22	1.44	1.61	0	0.05	-0.3	-0.85	-1.40
3 <sup>rd</sup>	1.8	0.12	0.38	0.59	0.59	0.59	0.46	0.35	0.25	0.15	0	-0.13	-0.37	-0.69	-1.07	-1.07	-1.52	-2.07	-2.55	-3.11	0	-0.04	0	0.08	0.14

Table ( 22 ) the fourth modification equipped with the third overarm

----- Initial position  
 \_\_\_\_\_ Modified rms.

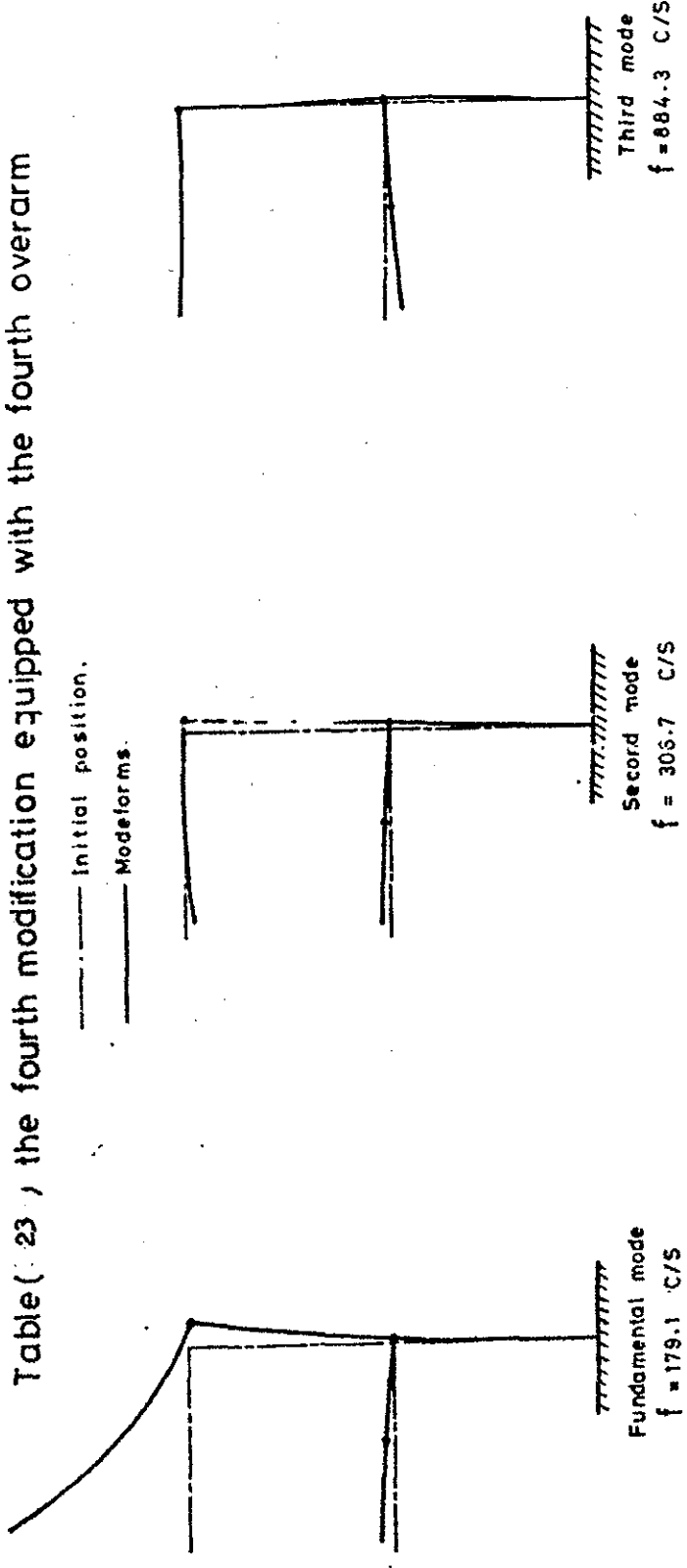
100



Mode no.		Deflections																									
		Beam 1					Beam 2					Beam 3					Beam 4					Beam 5					
		$X_1/L_1$					$X_2/L_2$					$X_3/L_3$					$X_4/L_4$					$X_5/L_5$					
$f$	$Z_1$	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	
1st	0.81	179.1	0	0.09	0.34	0.73	1.26	1.26	1.9	2.64	3.47	4.39	0	0.26	0.58	0.87	1.16	1.16	1.46	1.76	2.06	2.36	0	3.98	12.43	23.22	34.76
2nd	1.06	306.7	0	0.08	0.29	0.59	0.95	0.95	1.33	1.69	2.03	2.35	0	0.19	0.38	0.59	0.79	0.79	1	1.22	1.34	1.65	0	0.05	-0.29	-0.84	-1.44
3rd	1.8	866.3	0	0.12	0.38	0.58	0.59	0.59	0.46	0.35	0.25	0.15	0	-0.13	-0.37	-0.69	-1.07	-1.07	-1.52	-2.02	-2.56	-3.1	0	-0.04	0.01	0.09	0.14

Table ( 23 ) the fourth modification equipped with the fourth overarm

----- Initial position.  
 \_\_\_\_\_ Modeforms.

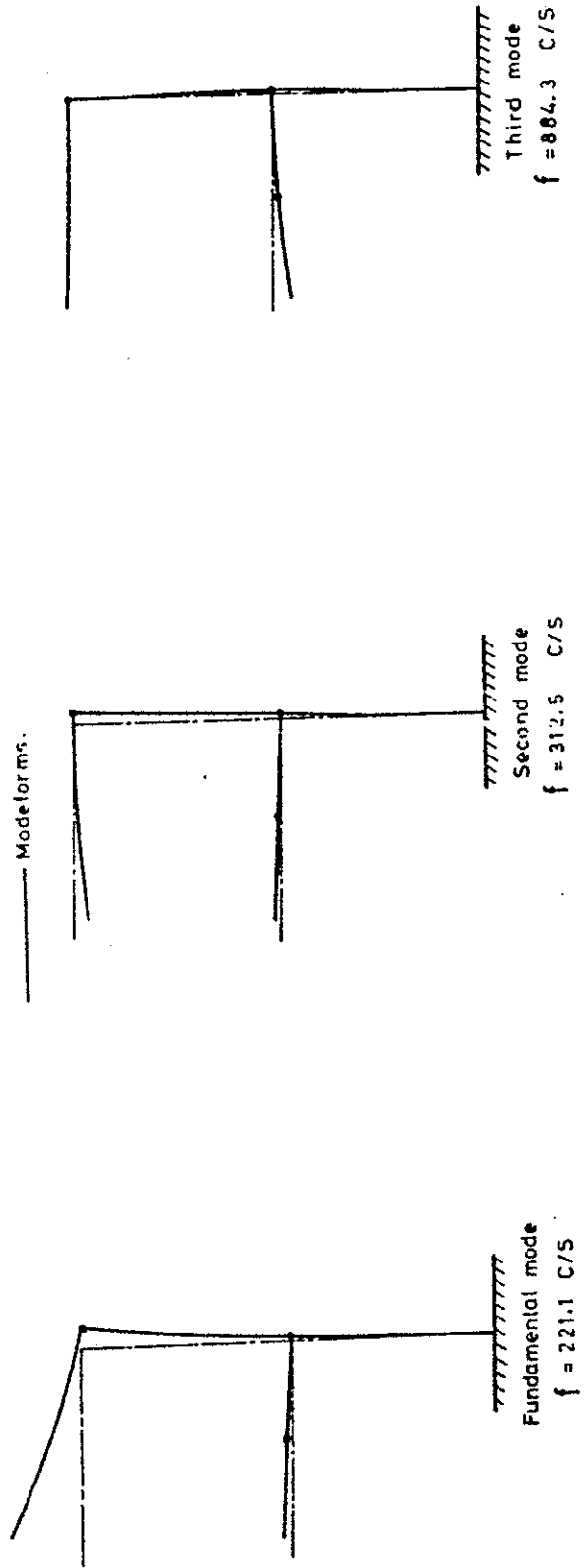


Mode no.	f c/s	Deflections																							
		Beam 1			Beam 2			Beam 3			Beam 4			Beam 5											
		$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$											
1 <sup>st</sup>	0.9	0.08	0.3	0.65	1.11	1.11	1.64	2.24	2.88	3.58	0	0.24	0.49	0.74	0.99	0.99	1.75	1.51	1.77	2.03	0	1.78	5.14	9.32	13.76
2 <sup>nd</sup>	1.07	0.07	0.28	0.57	0.92	0.92	1.29	1.63	1.94	2.22	0	0.18	0.37	0.56	0.76	0.76	0.96	1.17	1.38	1.59	0	0.04	-0.54	-1.3	-2.13
3 <sup>rd</sup>	1.8	0.12	0.38	0.58	0.58	0.45	0.35	0.25	0.15	0	-0.13	-0.37	-0.7	-1.08	-1.08	-1.08	-1.53	-2.04	-2.58	-3.13	0	-0.03	0.04	0.13	0.2

Table ( 24 ) the fourth modification equipped with the fifth overarm

----- Initial position

----- Modeforms.





Mode number	Deflection ratios																								
	Beam 1					Beam 2					Beam 3					Beam 4					Beam 5				
	$X_1/L_1$					$X_2/L_2$					$X_3/L_3$					$X_4/L_4$					$X_5/L_5$				
	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1
1 <sup>st</sup> mode	0	1.12	1.03	1.03	1.03	1.03	1.05	1.05	1.05	1.06	0	1.03	1.03	1.04	1.04	1.04	1.04	1.04	1.04	1.04	0	1.26	1.27	1.28	1.29
2 <sup>nd</sup> mode	0	1	1	1	1.01	1.01	1.02	1.02	1.03	1.03	0	1	1.02	1.01	1.02	1.02	1.02	1.02	1.01	1.02	0	4	-0.63	-0.05	0.23
3 <sup>rd</sup> mode	0	1	1	1	1	1	1	1	1	1	0	1	1	1	1	1	1	1	1	1	0	0.75	2	1	0.86

Table ( 25 ) Comparison between the fourth modification equipped with the first overarm with its basic model.

Mode number	Deflection ratios																								
	Beam 1					Beam 2					Beam 3					Beam 4					Beam 5				
	$X_1/L_1$					$X_2/L_2$					$X_3/L_3$					$X_4/L_4$					$X_5/L_5$				
	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1
1 <sup>st</sup> mode	0	1	0.91	0.89	0.88	0.88	0.88	0.86	0.84	0.83	0	0.65	0.66	0.66	0.66	0.66	0.86	0.86	0.86	0.86	0	0.46	0.42	0.41	0.41
2 <sup>nd</sup> mode	0	0.87	0.96	0.96	0.97	0.97	0.96	0.96	0.95	0.93	0	0.95	0.97	0.95	0.96	0.96	0.96	0.96	0.95	0.96	0	-0.8	2.04	1.62	1.52
3 <sup>rd</sup> mode	0	1	1	1	1	1	1	1	1	1	0	1	1	1	1	1	1.01	1.01	1.01	1.01	0	0.5	5	1.55	1.33

Table(25 ) Comparison between the fourth modification equipped with the second overarm with its basic model.

Mode number	Deflection ratios																			
	Beam 1			Beam 2			Beam 3			Beam 4			Beam 5							
	$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$							
0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	
1 <sup>st</sup> mode	0	1.12	1.03	1.01	1.02	1.02	1.03	1.04	1.04	0	1.03	1.03	1.03	1.03	1.03	1.03	1.03	1.03	1.03	1.03
2 <sup>nd</sup> mode	0	1	1	1	1	1	1	1	1	0	1	1	1	1	0.96	1.01	1	1	1	1
3 <sup>rd</sup> mode	0	1	1	1	1	1	1	1	1	0	1	1	1	1	1	1	1	0.99	0.95	0

Table( 27 ) Comparison between the fourth modification equipped with the third overarm with its basic model.

Mode number	Deflection ratios																			
	Beam 1			Beam 2			Beam 3			Beam 4			Beam 5							
	$X_1/L_1$			$X_2/L_2$			$X_3/L_3$			$X_4/L_4$			$X_5/L_5$							
0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	
1 <sup>st</sup> mode	0	1.12	1.03	1	1.01	1.01	1.02	1.02	1.02	0	1	1.02	1.01	1.01	1.01	1.01	1.01	1.01	1.01	1.01
2 <sup>nd</sup> mode	0	1	1	1	1	1	1	1	1	0	1	1	1	1	1	1	1	1	1	1
3 <sup>rd</sup> mode	0	1	1	1	1	1	1	1	1	0	1	1	1	1	1	1	1	1	1	0.93

Table( 28 ) Comparison between the fourth modification equipped with the fourth overarm with its basic model.

Mode number	Frequency ratios.	Deflection ratios																													
		Beam 1					Beam 2					Beam 3					Beam 4					Beam 5									
		$X_1/L_1$					$X_2/L_2$					$X_3/L_3$					$X_4/L_4$					$X_5/L_5$									
1 <sup>st</sup> mode	1.2	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1	0	0.25	0.5	0.75	1
2 <sup>nd</sup> mode	1.02	0	0.87	0.96	0.96	0.97	0.89	0.88	0.86	0.85	0.83	0	0.86	0.86	0.86	0.86	0.86	0.87	0.87	0.87	0.87	0	0.8	0.8	0.8	0.8	0	0.8	0.8	0.8	0.8
3 <sup>rd</sup> mode	1	0	1	1	1	0.98	0.98	0.98	0.96	1	1	0	1	1	1.01	1.01	1.01	1	1.01	1.01	1.01	0	0.75	4	1.44	1.33					

Table (. 29 ) Comparison between the fourth modification equipped with the fifth overarm with its basic model.

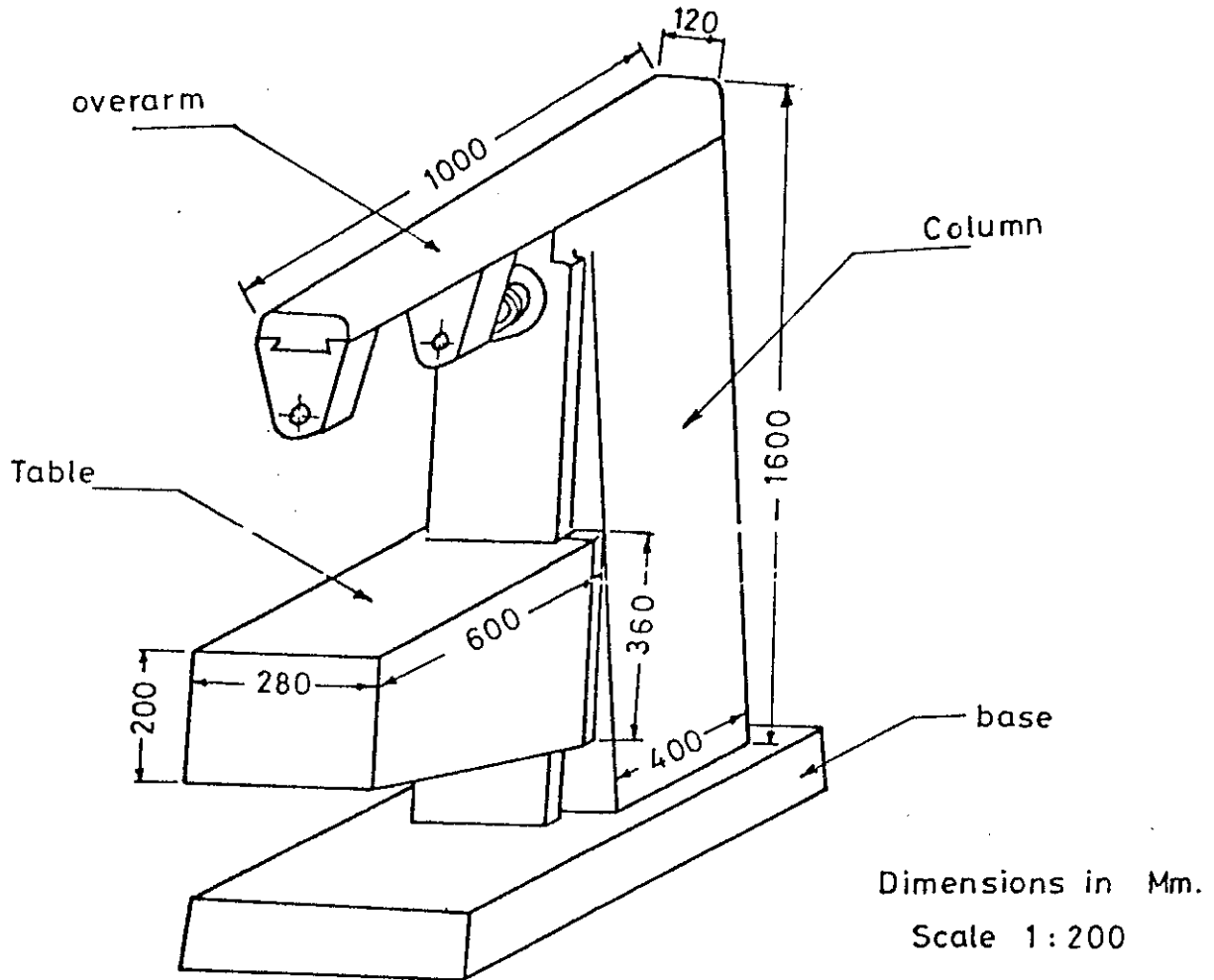


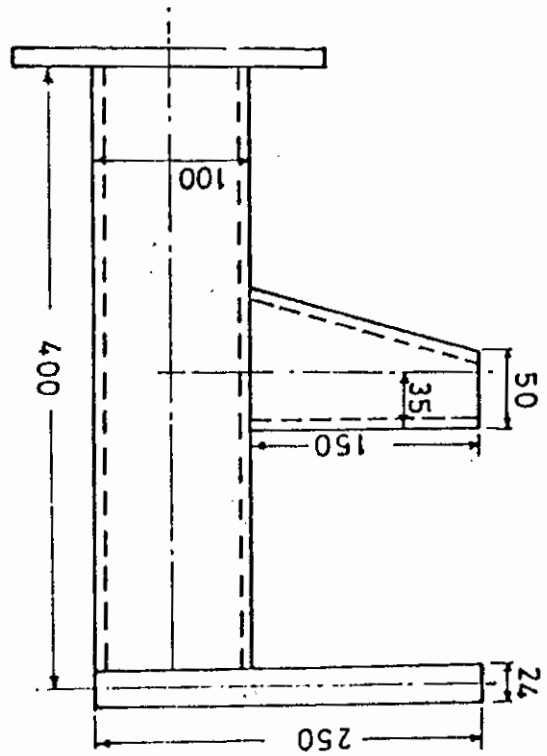
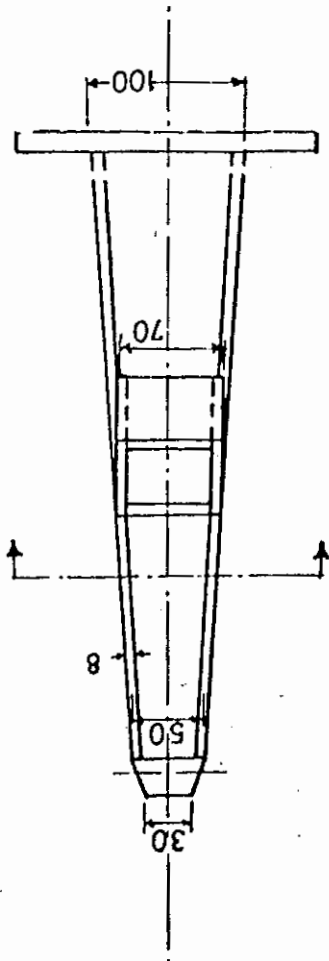
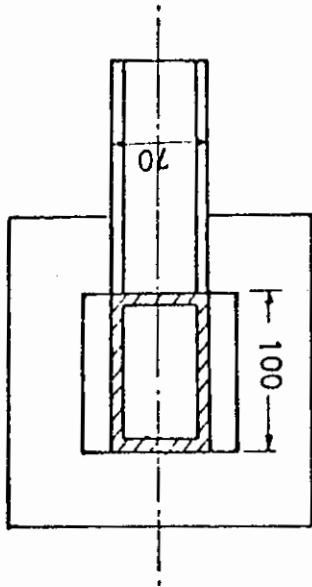
Fig. ( 1 )

Main parts and dimensions of the horizontal  
knee\_type milling machine.

The basic model of the machine.

Fig. (2)

Dimensions in Mm.  
Scale 1:50



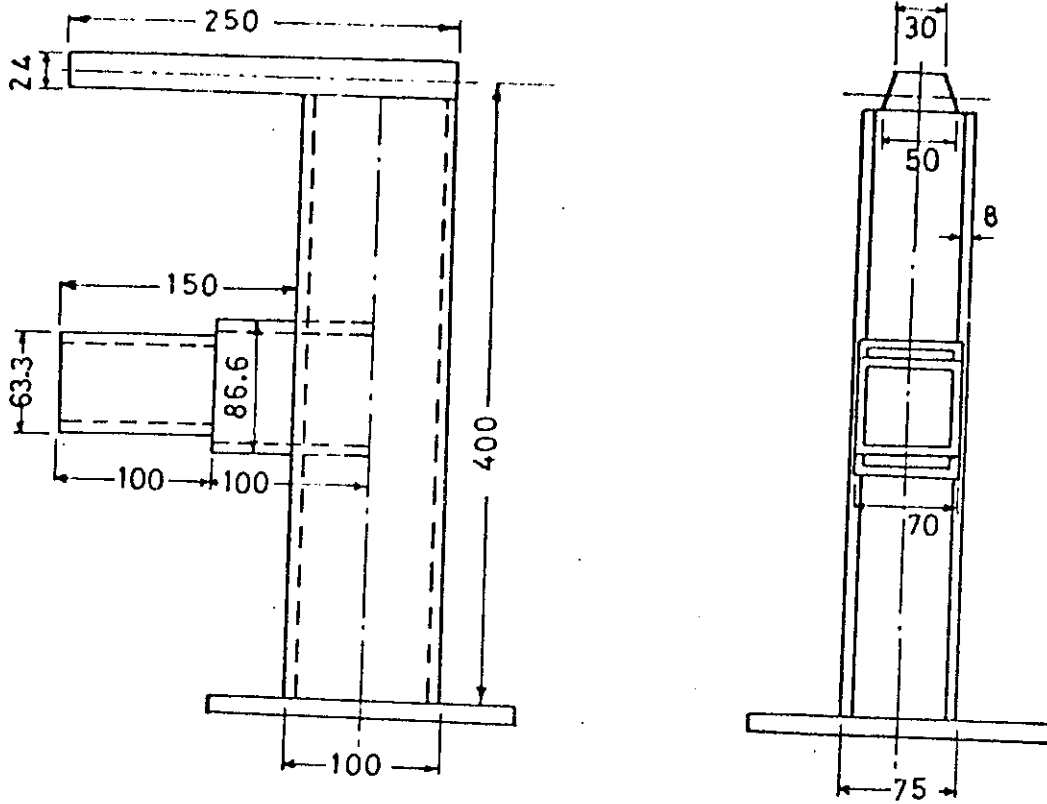


Fig. (3)

Idealized basic model

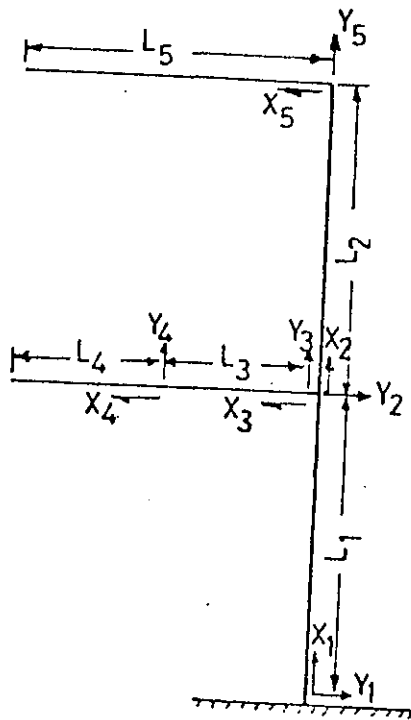


Fig. (4)

Mathematical model



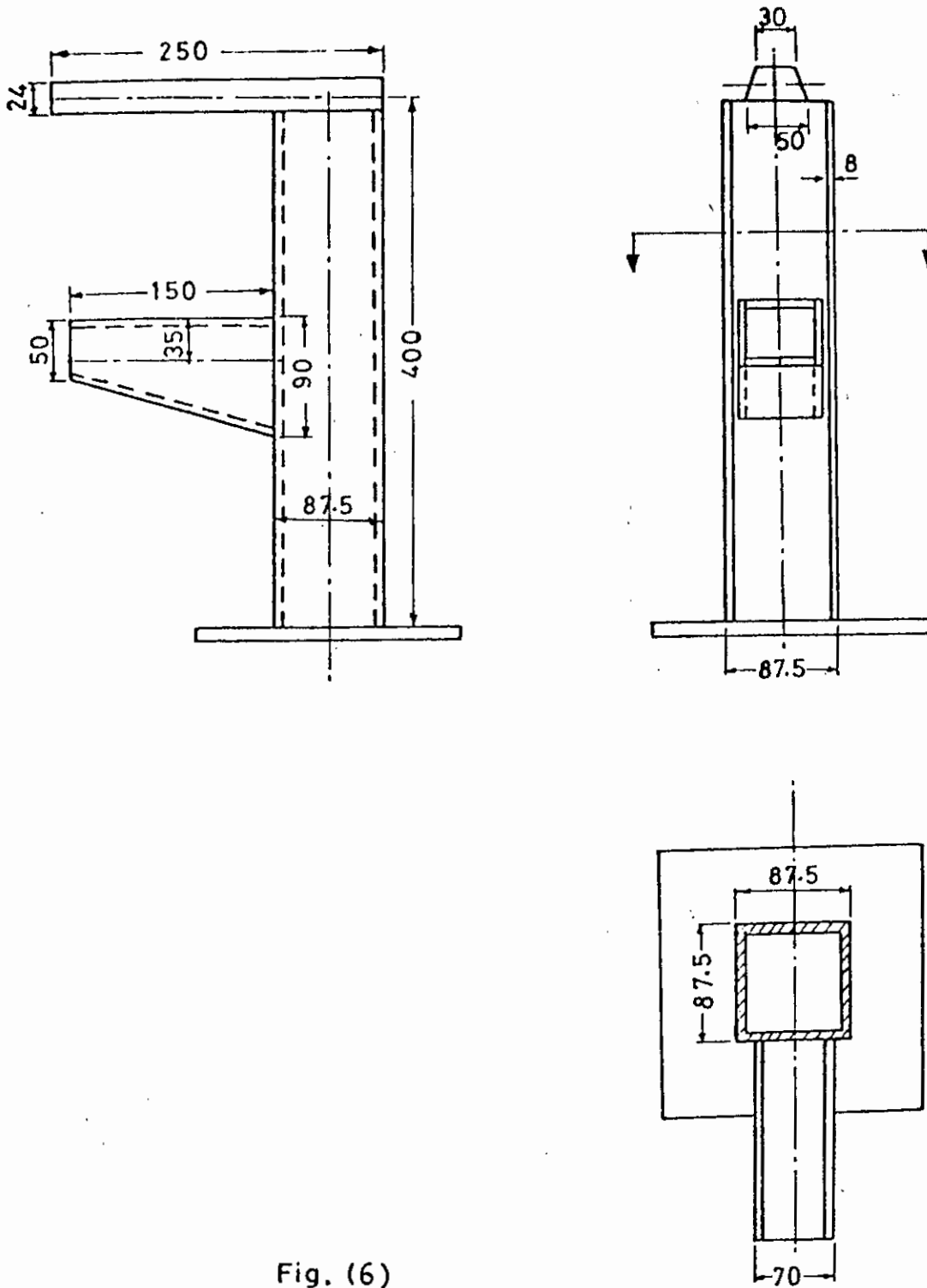


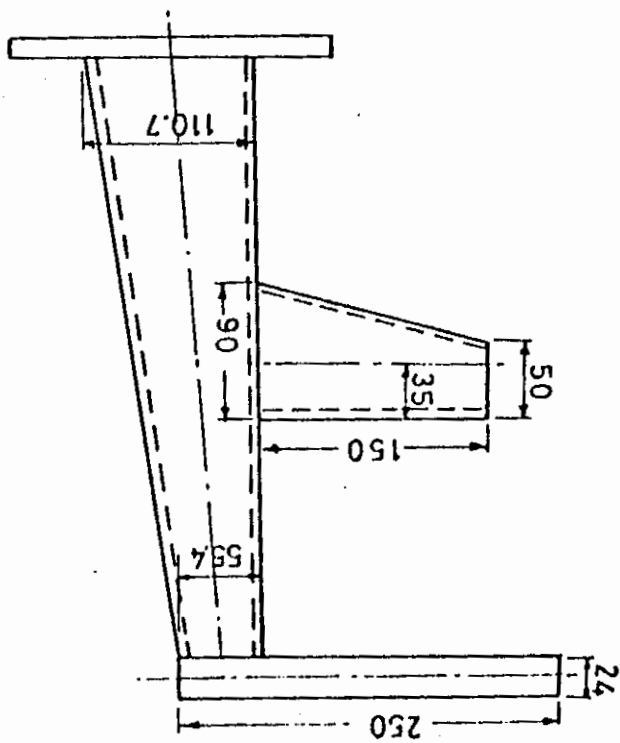
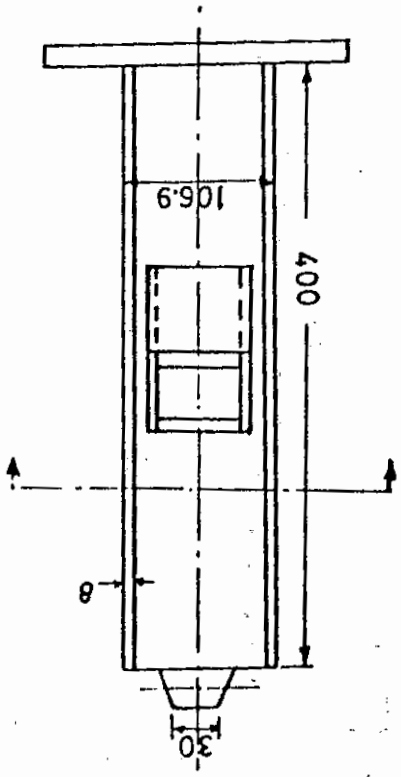
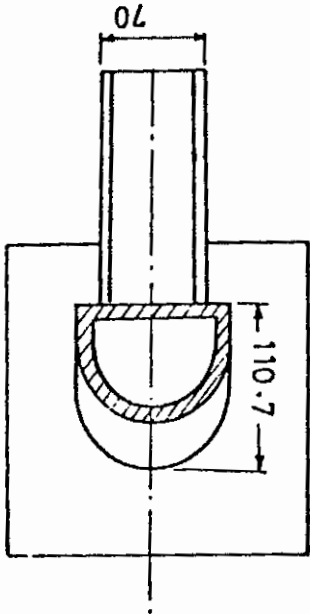
Fig. (6)

Model of the second modification



Model of the third modification.

Fig. (7)



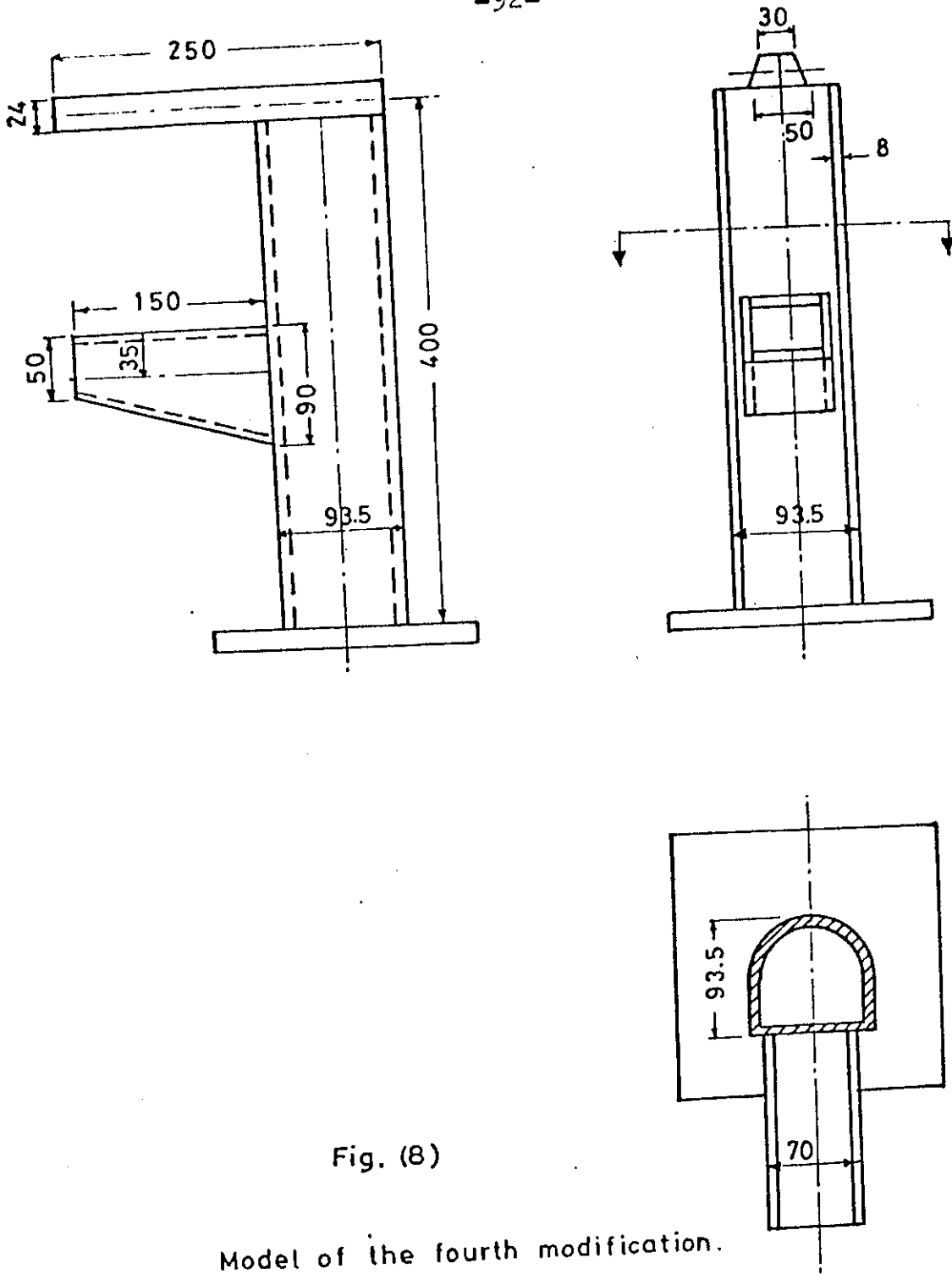


Fig. (8)

Model of the fourth modification.

The idealized models of the four suggested modification

Fig. ( 11 )

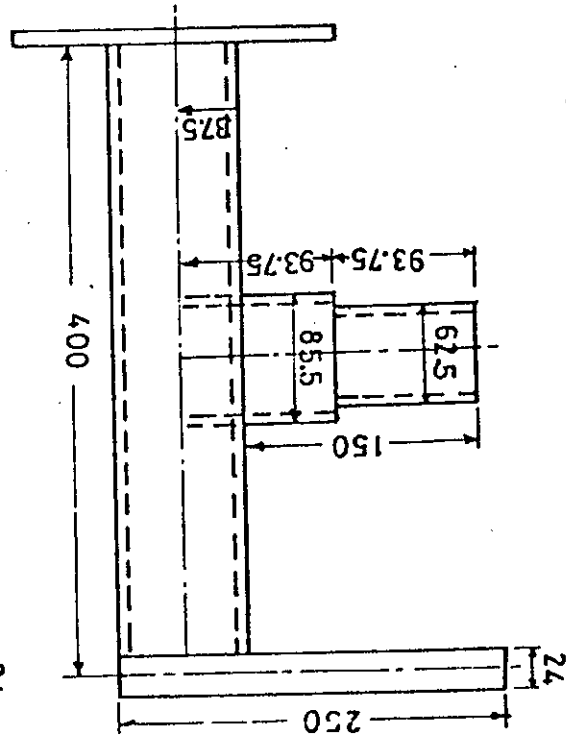


Fig. ( 12 )

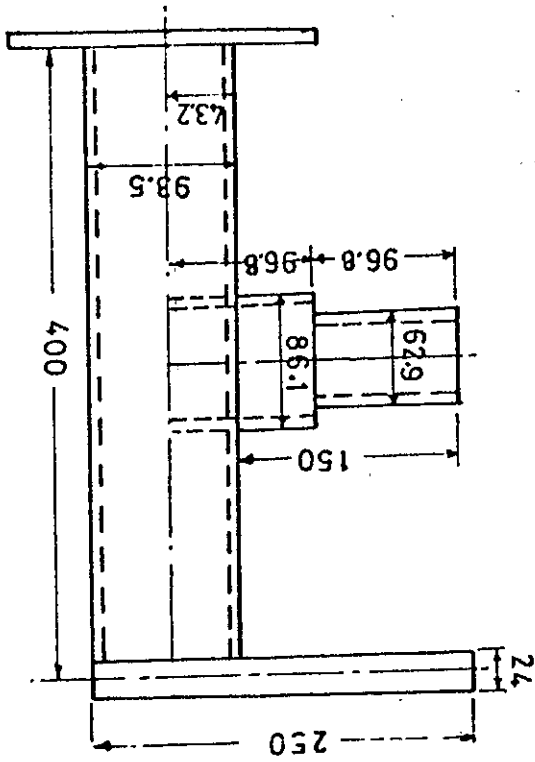


Fig. ( 9 )

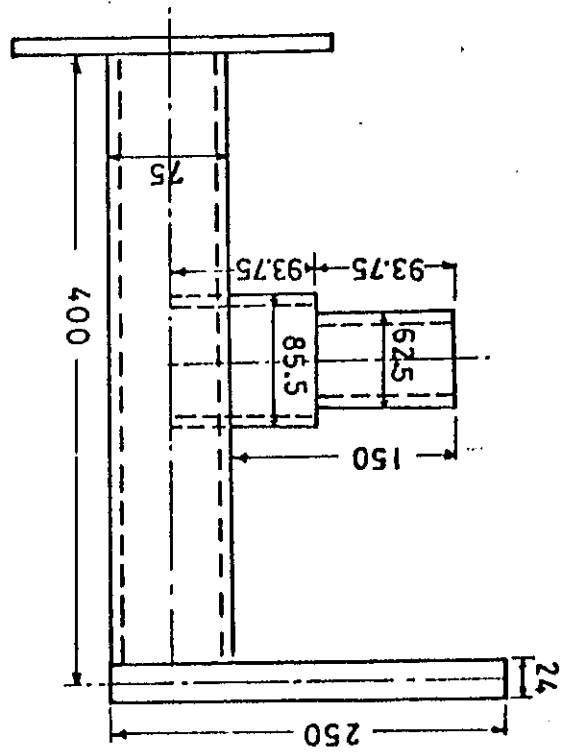
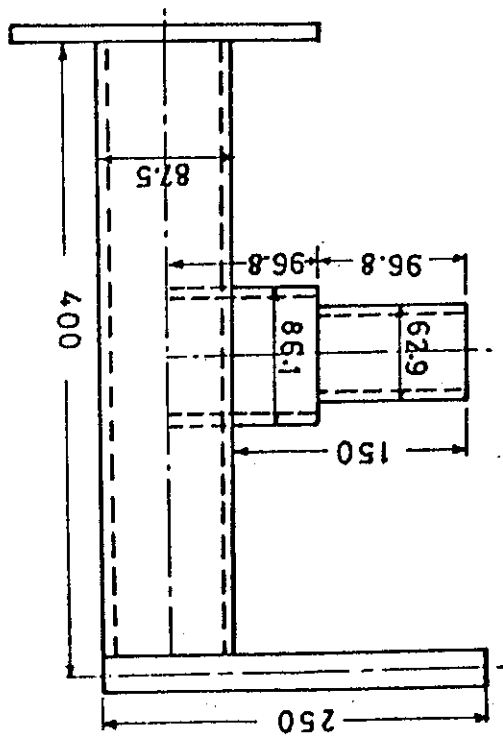
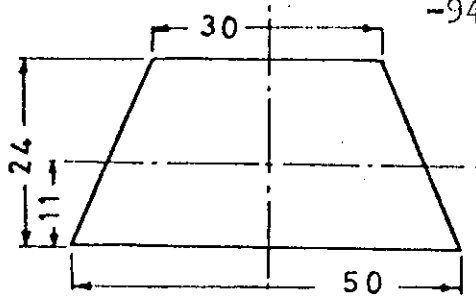
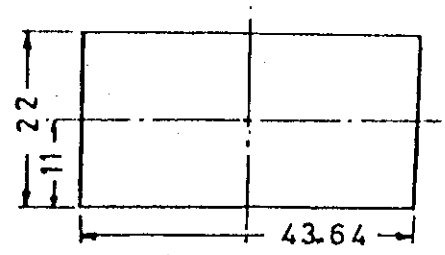


Fig. ( 10 )

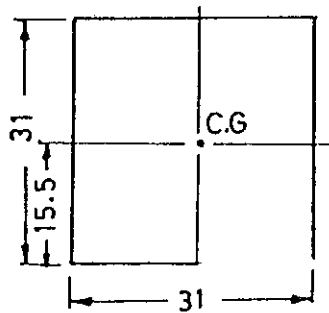




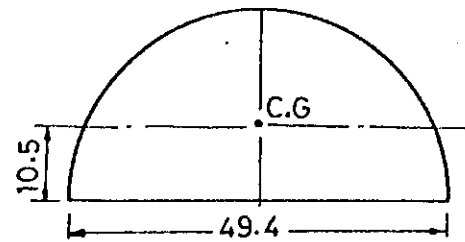
The basic cross-section  
(Trapezoid)



The first modification  
(Rectangle)



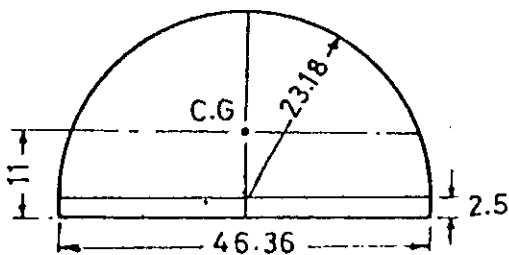
The second modification  
(Square)



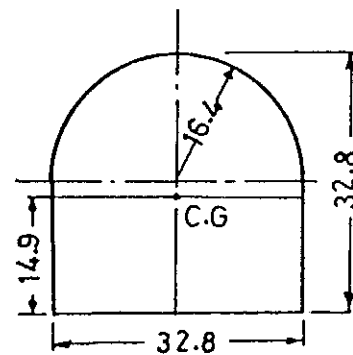
The third modification  
(Semi-Circle)

Dimensions in Mm.

Scale 1:1



The fourth modification  
(Semi-Circular on rectangular base)



The fifth modification  
(Semi-Circular tail)

Fig. ( 13 )

Different Cross-Sectional alternative shapes of the overarm.

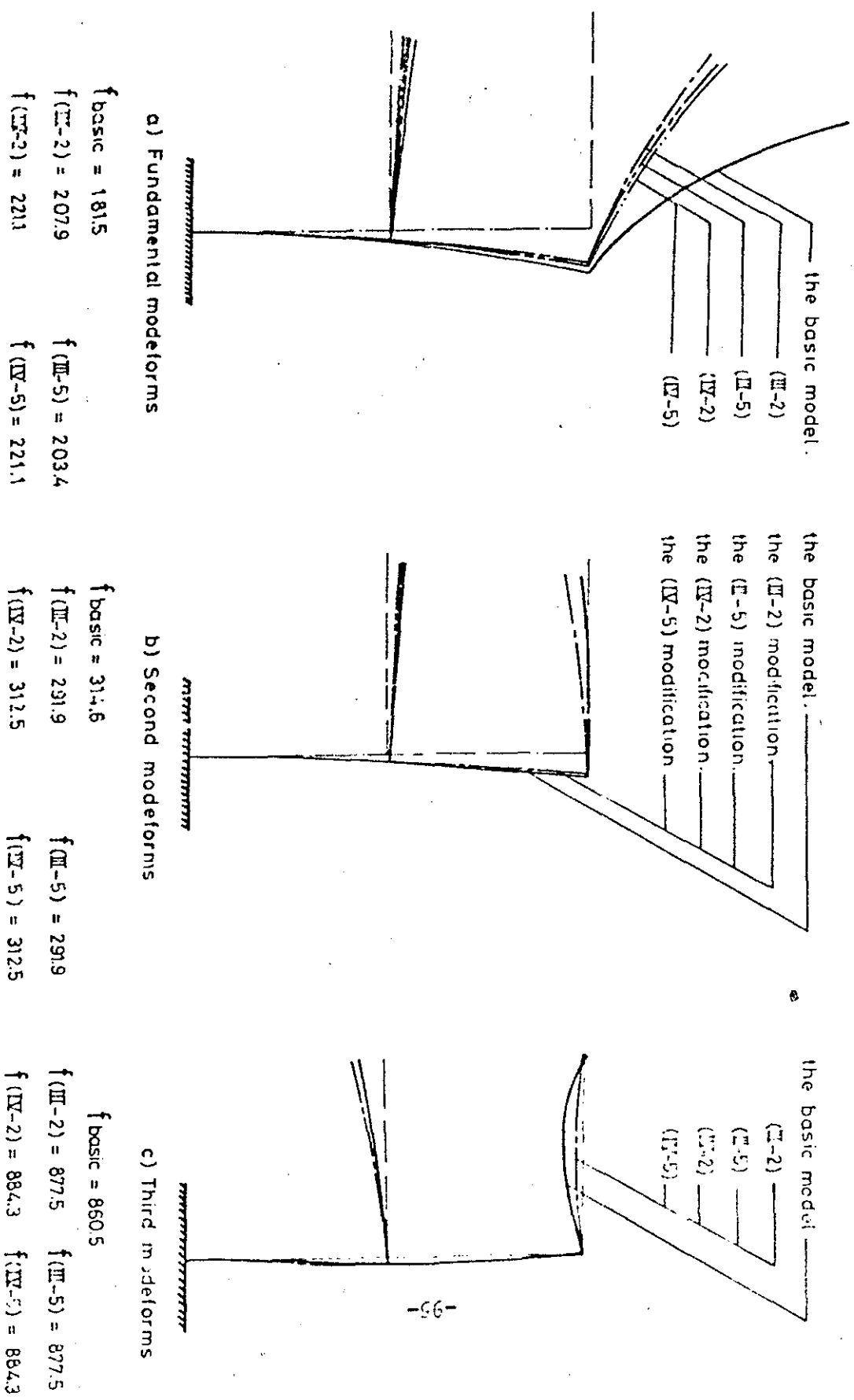


Fig. (14) Model forms of the basic model and the best modifications

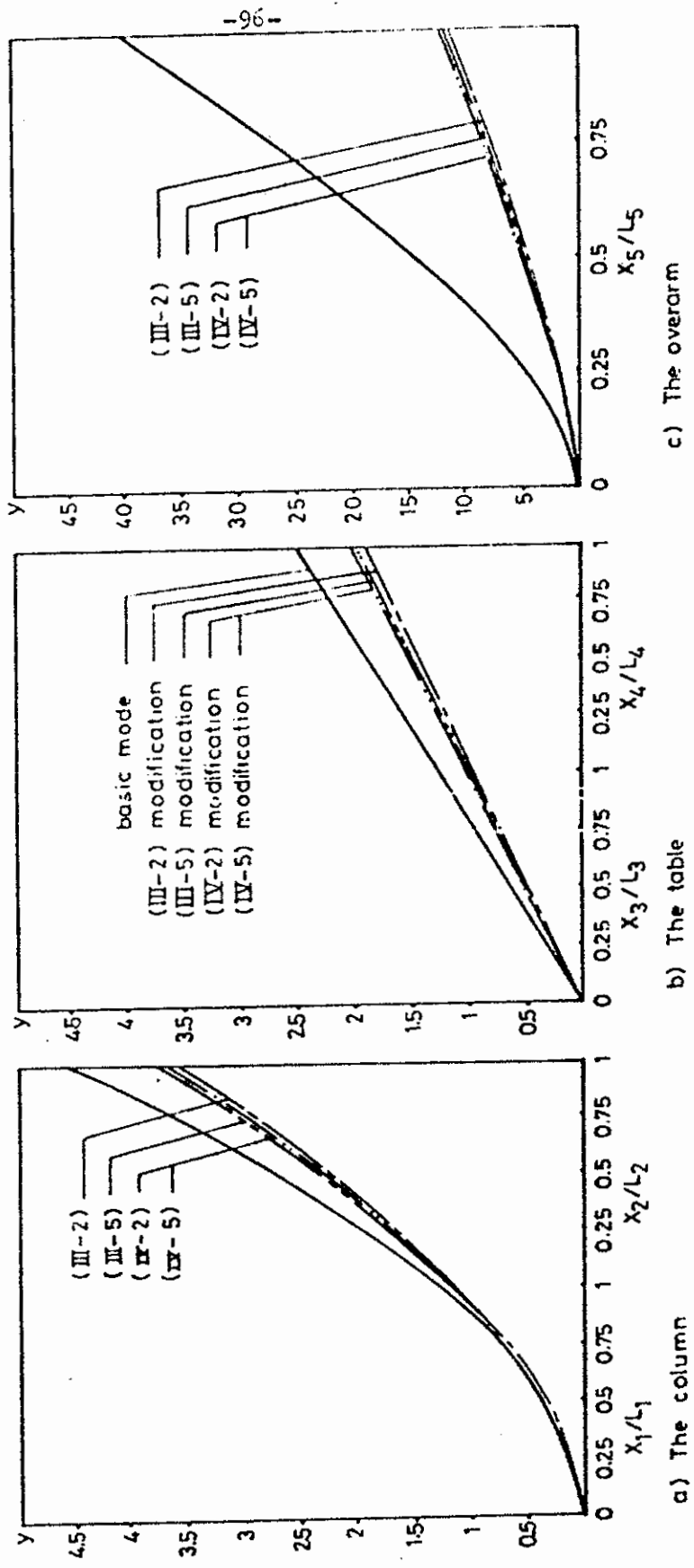


Fig.( 15 ) Deflections of the main parts of the basic model and best modifications

( First mode )

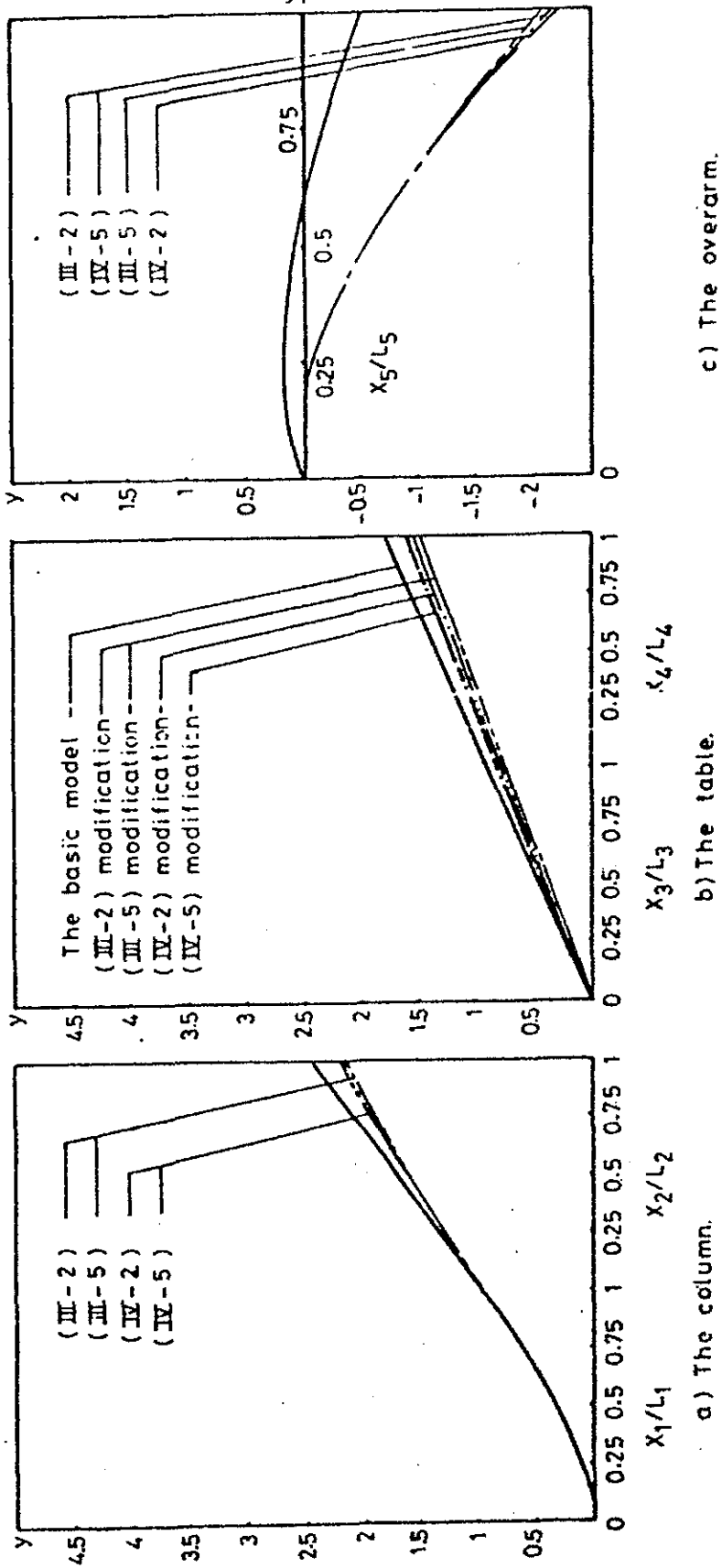
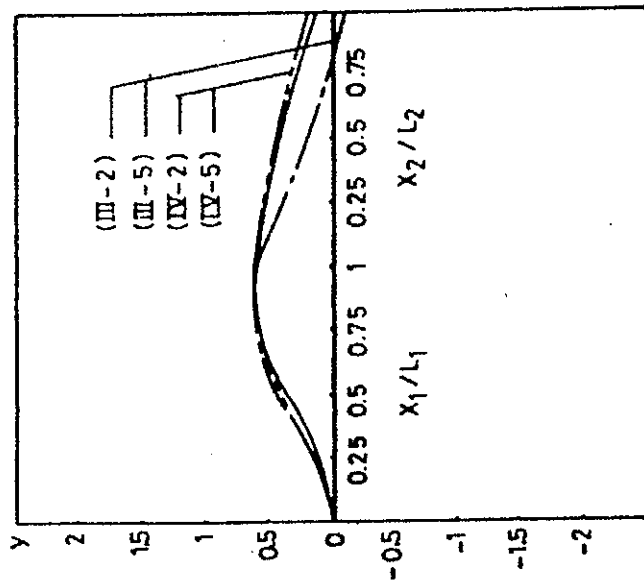
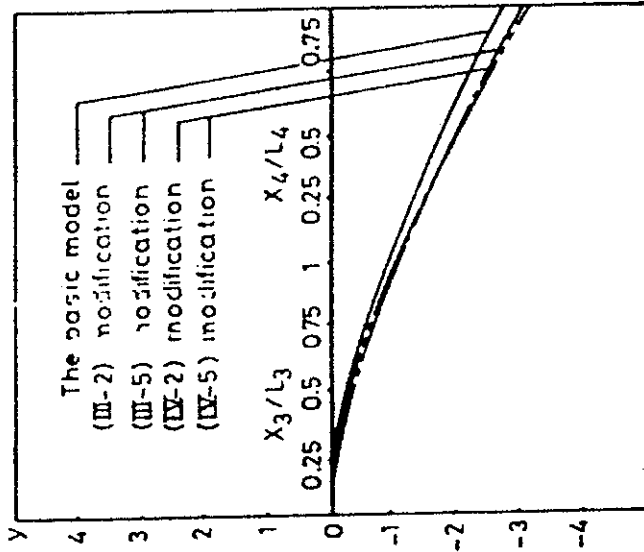


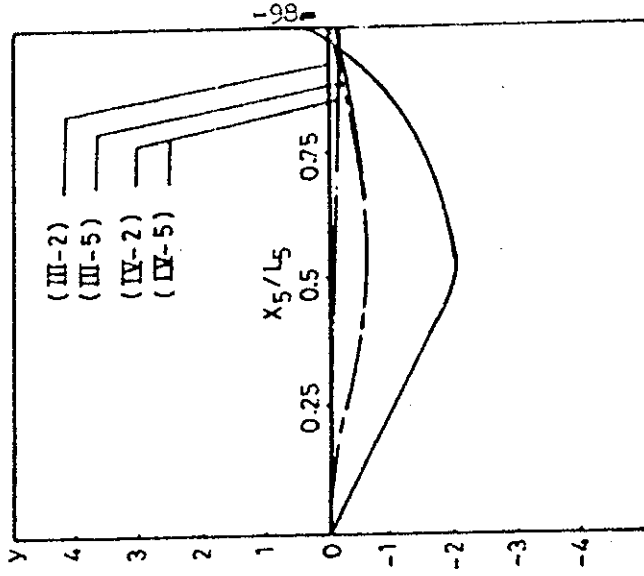
Fig. ( 16 ) Deflections of the main parts of the basic model and best modifications.  
( Second model ).



a) The column



b) The table



c) The overarm

Fig.(17) Deflections of the main part of the basic model and best modifications  
(Third mode)



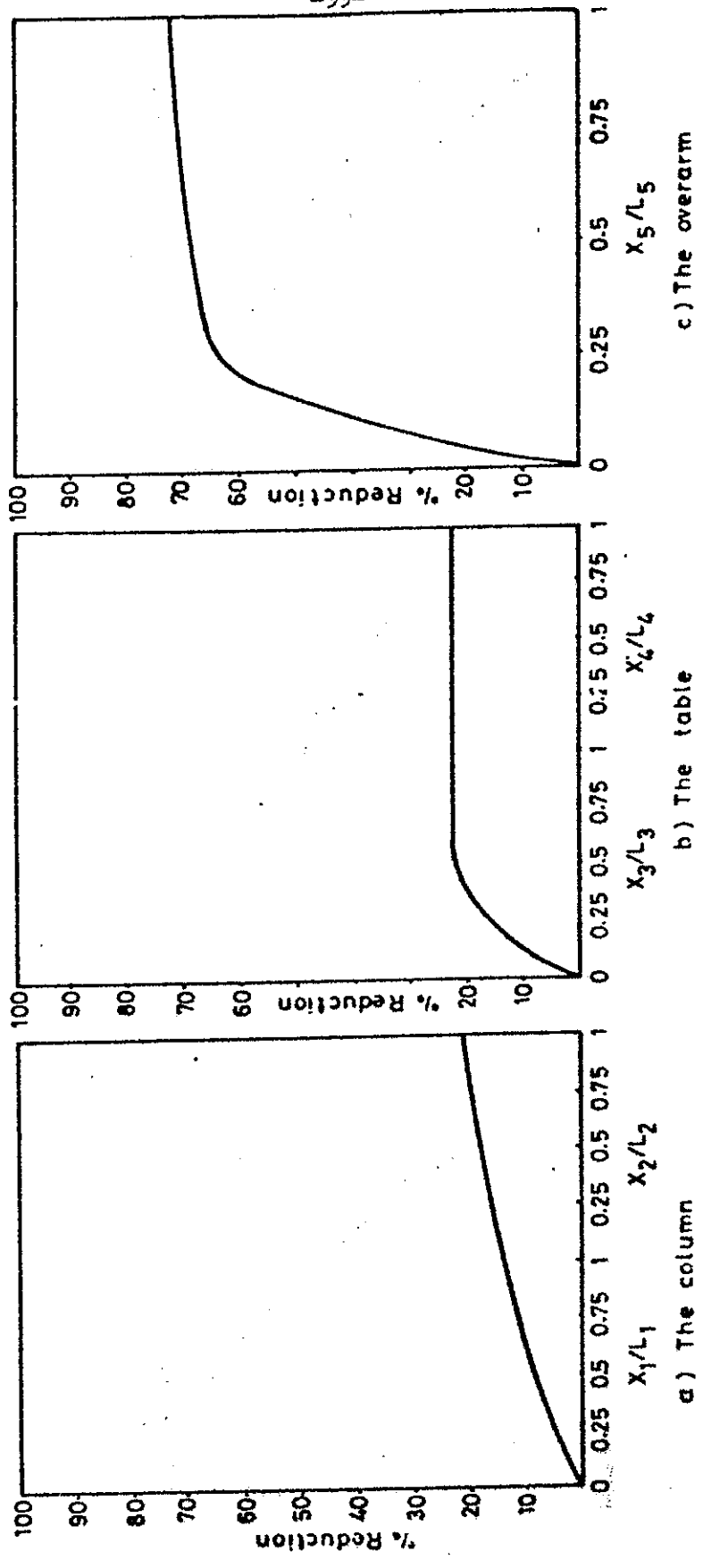


Fig ( 18 ) The percentage reduction in the deflection of the main parts of the model ( III - 2 ) as compared with the basic model

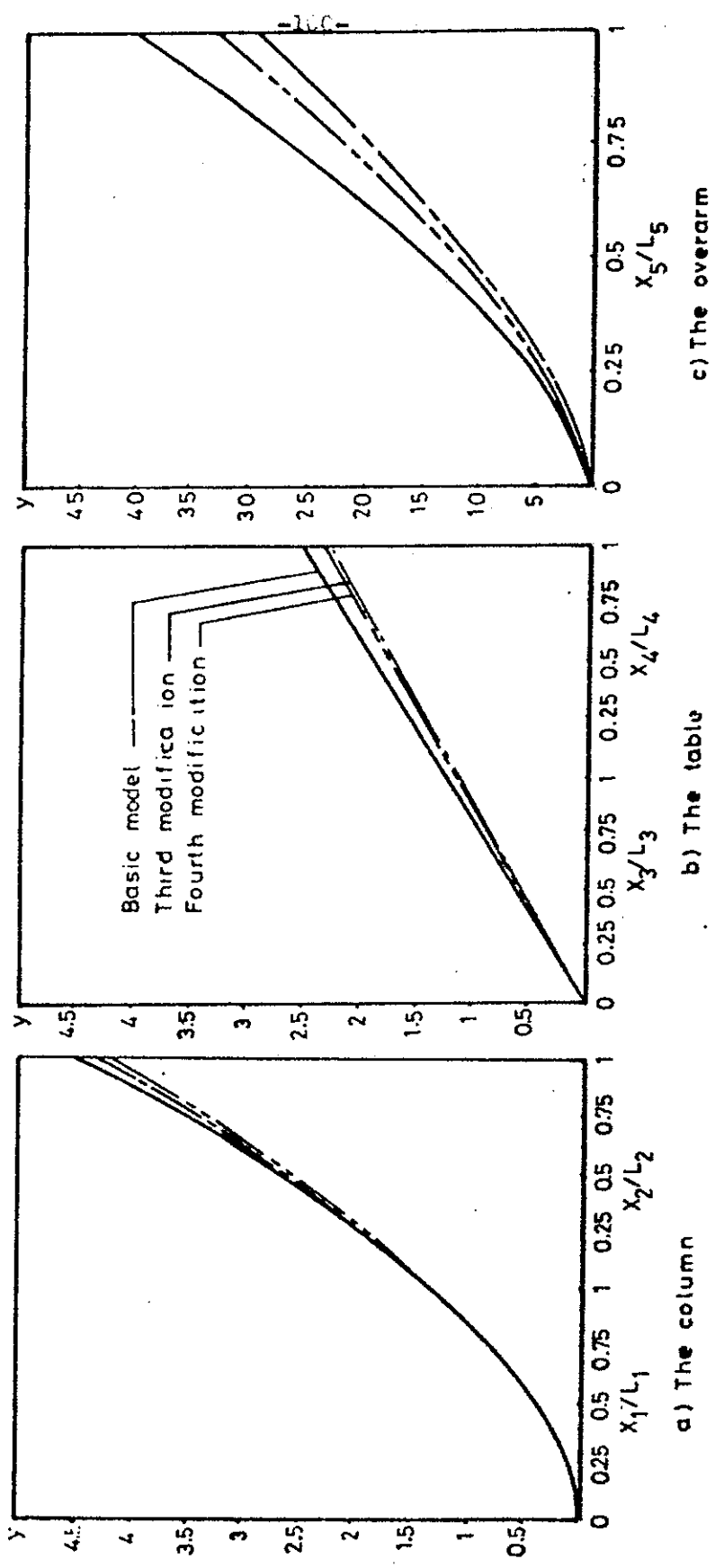
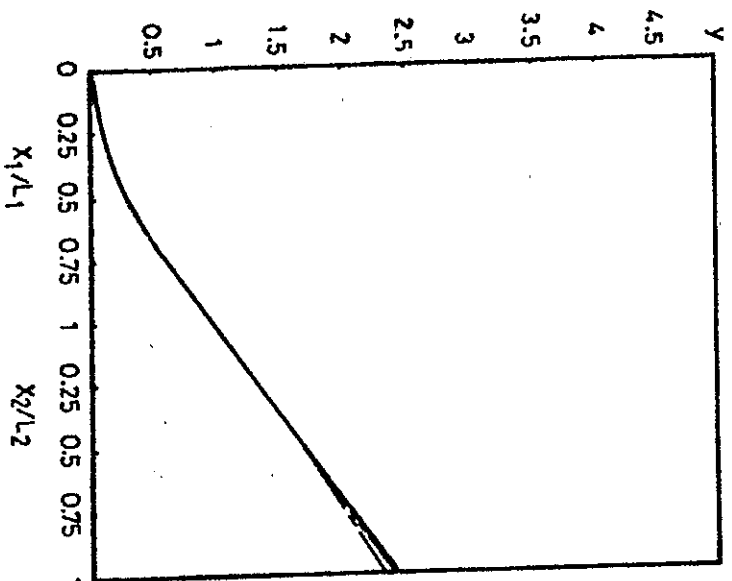
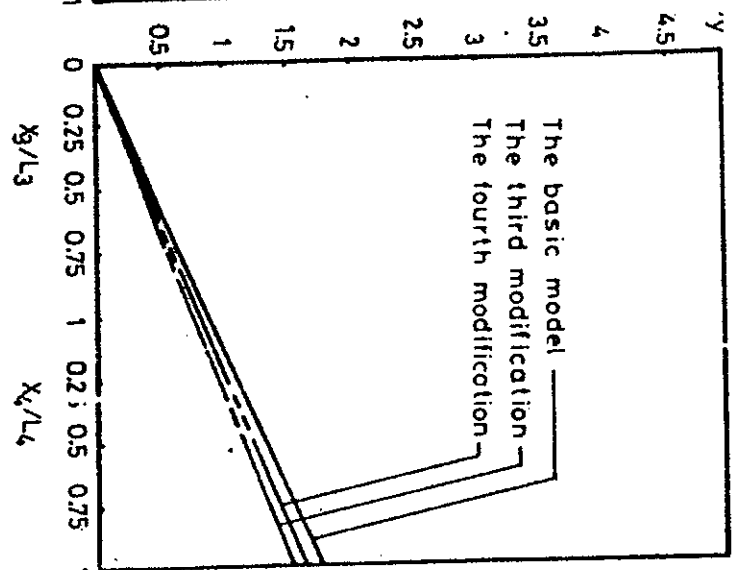


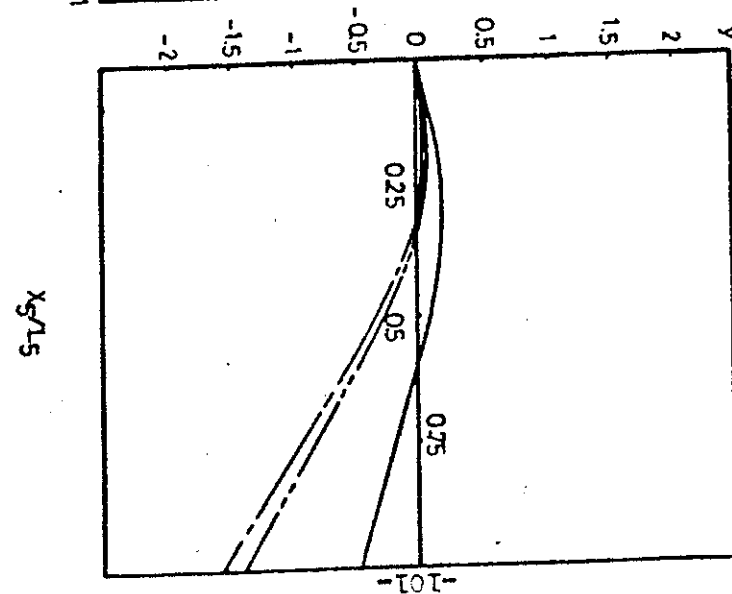
Fig. (19) Deflections of the main parts of the basic, third and fourth modifications  
( First mode )



a) The column



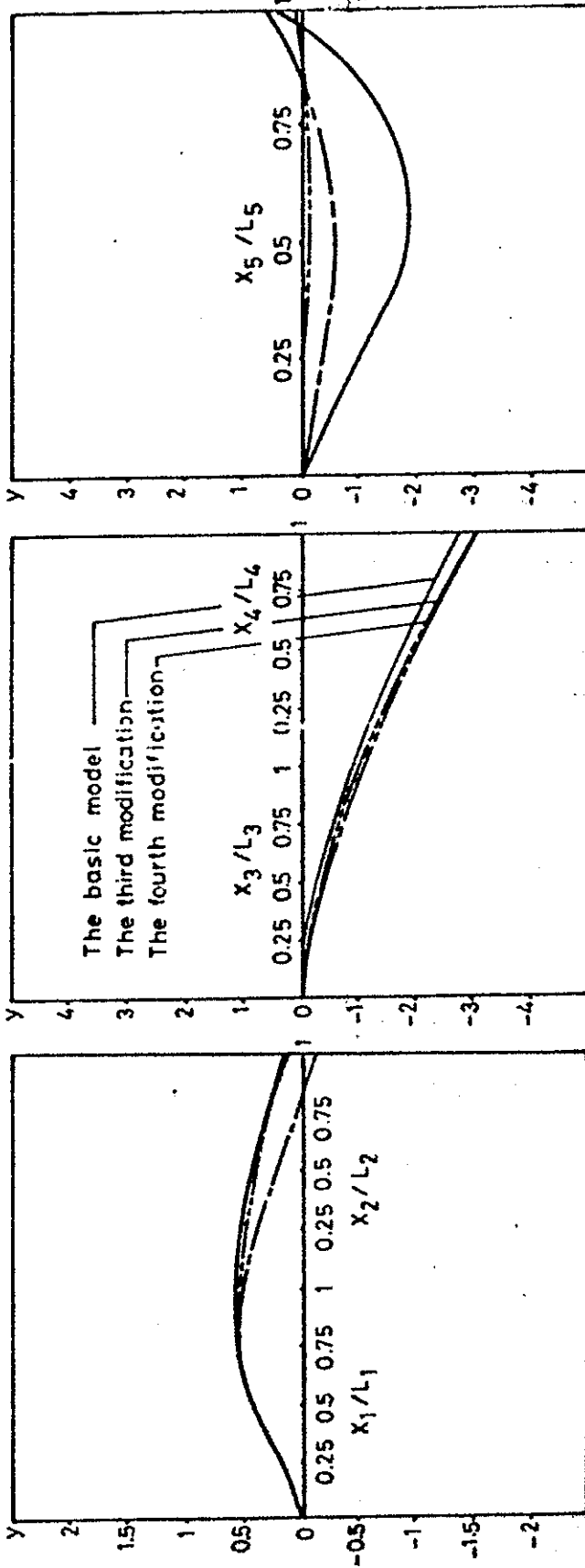
b) The table



c) The overarm

Fig. ( 20 ) Deflections of the main parts of the basic third and fourth modifications

( Second mode )



a) The column.

b) The table.

c) The overarm.

Fig. ( 21 ) Deflection of the main parts of basic, third and fourth modifications.  
( third mode ).

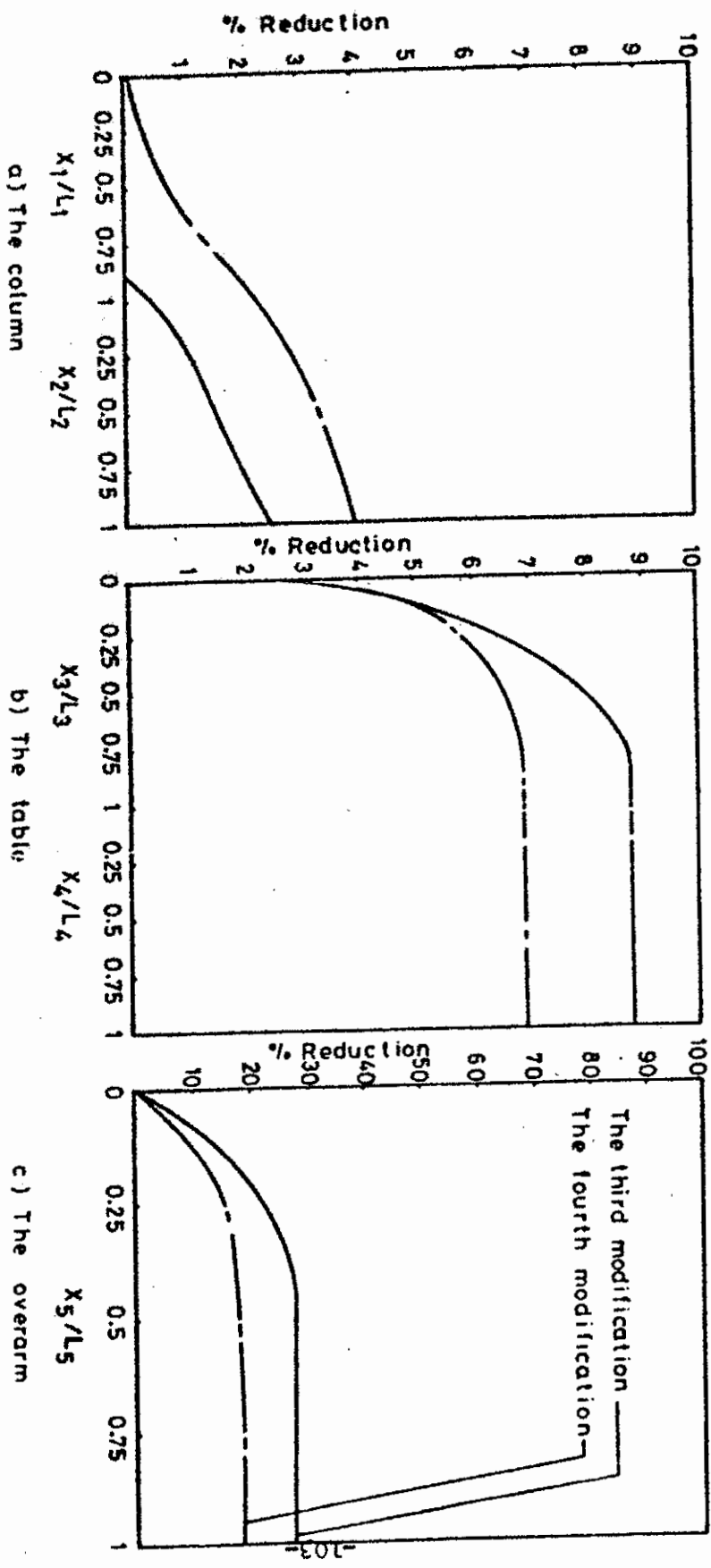


Fig. ( 22. ) The percentage reduction in the deflection of the main parts of the third and modifications as compared with the basic model.

MATHEMATICAL SIMULATION OF  
MACHINE TOOL STRUCTURES

التوصيف الرياضى لهياكل آلات الانتاج

أستاذ دكتور / عبدالهادى ناعمر • أستاذ دكتور / حسن رجب السيد  
دكتور / سعاد محمد سراج • مهندس / سالم محمد

هيكـل آلة الانتاج هو الجزء الرئيسى لها حيث يساهم فى تحديد حالتها  
الديناميكية . لهذا السبب كان من الضرورى دراسة أشكال تصميمية مختلفة  
للهيكل بمساعدة التوصيف الرياضى للهيكل .

فى هذا العمل فحصت الخصائص الديناميكية لهيكل فريزة أفقية على أساس  
طريقة نظرية العتب مختلا على النموذج الرياضى للهيكل . ومساعدة النموذج  
الرياضى المقدم درس وقورن ١٣ نموذج مقترح يمثلوا حلولاً تصميمية لها نفس  
الوزن ، الأبعاد ، السمك والكتلة .

الدراسة المقدمة بينت امكان توقع الحالة الديناميكية لمعظم الهياكل الملائمة  
وبالتالى امكان حل التحسينات الممكنة لها .